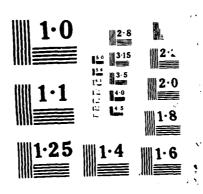
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SINGLE PHASE LIQUID IMMERSION COOLING OF DISCRETE HEAT SOURCES IN A VERTICAL CHANNEL

by

Sherrill John Hazard, III

December 1987

Thesis Advisor:

Yogendra Joshi

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#19 - ABSTRACT - (CONTINUED)

Visual results indicate two distinct flow regions. Far away from the components, a natural convection boundary layer flow was observed. Near the components, the flow was modified by the protrusions. As the component heat input increased, more pronounced three dimensional effects were noticed. Temperature measurements indicate that as the modified Grashof numbers increased, the nondimensional temperatures decreased for each component. Also, the difference in the nondimensional temperatures for various components decreased with increasing modified Grashof numbers.

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Single Phase Liquid Immersion Cooling of Discrete Heat Sources in a Vertical Channel

by

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Lieutenant, United States Navy
B.S., University of Maine at Orono, 1980

Submitted in partial fulfillment of the requirements for the degree of

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ABSTRACT

liquid cooling Natural convection of simulated electronic components was investigated. A single column of eight protruding components was constructed using foil heaters mounted on the back of stainless steel rectangular These components were attached to a vertical plexiglas wall to simulate a column of 20 pin DIP's. channel was formed by placing a smooth movable shrouding wall parallel to the test surface. The test surface and the shrouding wall were placed in a water immersion bath. Flow visualization was accomplished using a laser generated plane of light to illuminate suspended particles. Photographs were taken of the flow at the test surface mid-plane for four different power settings at each of three different channel widths. A nondimensional temperature and a modified Grashof number for each heated protrusion at each input power setting and channel width were determined. Visual results indicate two distinct flow regions. Far away from the components, a natural convection boundary layer flow was observed. Near the components, the flow was modified by the protrusions. As the component heat input increased, more pronounced three dimensional effects noticed. were Temperature measurements indicate that as the modified Grashof numbers increased, the nondimensional temperatures

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THESIS DISCLAIMER

The reader is cautioned that computer programs developed in this research may not have been exercised for all cases of interest. While every effort has been made, within the time available, to ensure that the programs are free of computational and logic errors, they cannot be considered validated. Any application of these programs without additional verification is at the risk of the user.

TABLE OF CONTENTS

I.	INT	RODUCTION	1
	A.	STATEMENT OF PROBLEM	1
	в.	IMMERSION COOLING: ANALYTICAL AND EXPERIMENTAL STUDIES	2
	c.	OBJECTIVES	6
II.	EXP	ERIMENT	7
	A.	GENERAL DESIGN CONSIDERATIONS	7
		1. Heater Block Dimensions	8
		2. Heating Element	8
		3. Heater Location	9
		4. Visualization Technique	9
		5. Thermocouple Design and Placement	9
		6. Other Considerations	10
	в.	COMPONENTS	10
		1. Heater	10
		2. Test Surface and the Shrouding Wall	12
		3. Test Surface Back Containment	12
		4. Test Surface Support	13
		5. Immersion Tank	13
		6. Immersion Bath Filtration and Purification	13
	c.	ASSEMBLY	14
	D.	INSTRUMENTATION	15
		1. Power to the Heater	15

		2.	Temperature Measurement	16
III.	EXF	ERIME	ENTAL PROCEDURE	18
	A.	APP	ARATUS PREPARATION	18
	В.	TEST	PROCEDURE	19
		1.	Initial Instrument Settings	19
		2.	Instrument Readings	20
		3.	Photographic Technique	21
		4.	Experiment Completion	21
		5.	Channel Width	22
	c.	DATA	A ANALYSIS	22
IV.	RES	ULTS		28
	A.	FLOV	V VISUALIZATIONS	28
	в.	QUA	NTITATIVE	30
٧	CON	CLUSI	CONS	33
vI.	REC	OMMEN	NDATIONS	34
	A.	IMPF	ROVEMENT TO EXPERIMENT	34
		1.	Apparatus	34
		2.	Data Acquisition	35
	В.	ADDI	TTIONAL EXPERIMENTAL WORK	35
APPENI	XIC	A: 5	SAMPLE CALCULATIONS	36
APPENI	XIC	B: [JNCERTAINTY ANALYSIS	38
APPEN	XIC	C: 1	TABULAR DATA	42
APPENI	XIC	D: 5	SOFTWARE	78
APPENI	XIC	E: 1	TABULAR RESULTS	86
APPENI	хтс	F: F	TGURES	93

LIST OF	REFERENCES		140
INITIAL	DISTRIBUTIO	N LIST	142

LIST OF TABLES

I.	PHOTOGRAPH EXPOSURE VARIATIONS	21
II.	PROPERTIES OF WATER	24
III.	PHYSICAL CONSTANTS	25
IV.	THERMAL CONDUCTIVITY OF MATERIALS	25
7.7	IINCEDTATINTY WADTABLES	2.0

LIST OF FIGURES

1.	Temperature versus Heat Flux for Various Phenomena	93
2.	Assembled Test Surface and Shrouding Wall	94
3.	Mounted Foil Heater	94
4.	Mounted Thermocouple	95
5.	System Configuration	96
6.	20 Pin DIP and Chip Comparison, Top View	97
7.	20 Pin DIP and Chip Comparison, End View	97
8.	Laser and Cylindrical Lens	98
9.	Laser and Camera Orientation	99
10.	Block with Grooves	100
11	Mounted Foil Heater, End Measured	100
12.	Mounted Foil Heater, Length Measured	101
13.	Heater Block Schematic	102
14.	Foil Heater Schematic	103
15.	Power Lead Attachment	104
16.	Slot and Holes in Test Surface	105
17.	Containment Back Schematic	106
18.	Test Surface and Shrouding Wall Support Bracket	107
19.	Immersion Tank	108
20.	Filtration and Purification System	109
21.	Mounting the Heater Assemblies	110
22.	Mounted Heater Assemblies	111
23.	Close-Up of Mounted Heater Assemblies	112

24.	Thermo	ocouple Connection Schematic	113
25.		Visualization Photographs for the	114
26.	Flow '73.81	Visualization Photographs for the mm Spacing	115
27.	Flow 11.91	Visualization Photographs for the 3 mm Spacing	116
28.		s Test Surface Flow Visualization graph	117
29.		Number vs. Excess Temperature (Front Runs 1-4	118
30.		Number vs. Excess Temperature (Front Runs 5-8	119
31.		Number vs. Excess Temperature (Front Runs 9-12	120
32.		Number vs. Excess Temperature (Right Runs 1-4	121
		Number vs. Excess Temperature (Right Runs 5-8	122
34.		Number vs. Excess Temperature (Right Runs 9-12	123
35.	Block Face)	Number vs. Excess Temperature (Left Runs 1-4	124
36.		Number vs. Excess Temperature (Left Runs 5-8	125
37.		Number vs. Excess Temperature (Left Runs 9-12	126
38.		Number vs. Excess Temperature (Top Runs 1-4	127
39.		Number vs. Excess Temperature (Top Runs 5-8	128
40.		Number vs. Excess Temperature (Top Runs 9-12	129
41.		Number vs. Excess Temperature (Bottom Runs 1-4	130

42.	Block Number vs. Excess Temperature (Bottom Face) Runs 5-8	131
43.	Block Number vs. Excess Temperature (Bottom Face) Runs 9-12	132
44.	Block Number vs. Excess Temperature (Heater) Runs 1-4	133
45.	Block Number vs. Excess Temperature (Heater) Runs 5-8	134
46.	Block Number vs. Excess Temperature (Heater) Runs 9-12	135
47.	Block Number vs. Excess Temperature (Comparison of Front Face and a Flat Plate with Constant Heat Flux)	136
48.	Modified Grashof Number vs. Nondimensional Temperature Runs 1-4	137
49.	Modified Grashof Number vs. Nondimensional Temperature Runs 5-8	138
50.	Modified Grashof Number vs. Nondimensional Temperature Runs 9-12	139

NOMENCLATURE

<u>Symbol</u>	<u>Description</u>	<u>Units</u>
A	Area	m ²
g	Acceleration due to gravity	m/s ²
Gr*	Modified Grashof Number	Dimensionless
k	Thermal conductivity	W/m-°C
kf	Fluid thermal conductivity	W/m-°C
k_{PG}	Thermal conductivity of plexiglas	W/m-°C
$k_{\mathbf{R}}$	Thermal conductivity of foam rubber insulation	W/m-°C
L	Characteristic length	m
QCOND	Energy loss via conduction through the back of the test surface	W
QCONV	Energy convected into fluid	W
Q_{IN}	Energy into foil heater	W
R	Resistance of precision resistor	Ω
RA	Equivalent thermal resistance to conduction through plexiglas test surface	°C/w
R _B	Equivalent thermal resistance to conduction through foam rubber insulation	°c/w
TAVG	Average block surface temperature	°c
TINF	Ambient temperature	°c
TB	Bottom surface temperature of the block	°C

$\mathtt{T}_{\mathbf{F}}$	Front surface temperature of the block	°C
TA	Heater temperature	°C
TL	Left surface temperature of the block	°c
T_{R}	Right surface temperature of the block	°c
${\tt T}_{{\tt T}}$	Top surface temperature of the block	°C
v_{H}	Voltage across heater	Volts
$v_{\mathbf{T}}$	Input voltage	Volts
δ	Uncertainty of variable	Various
β	Coefficient of expansion	1/°C
ν	Kinematic viscosity	m^2/s

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I. INTRODUCTION

A. STATEMENT OF PROBLEM

From the first electronic digital computer, ENIAC (1946), which used vacuum tubes as its basic logic elements, to the present use of integrated circuits, rapid advancements in the miniaturization of electronic components is well documented [Refs. 1,2]. This "boom" in technology has brought us from the small scale integration (SSI) device to the ultra large scale integration (ULSI) device in just 25 years [Ref. 3]. With the 1970 advent of the first one kilobit RAM semiconductor device, the number of memory cells contained on a single device has grown to the familiar 64K and more recently the 256K and 512K devices.

While the component density per chip has increased significantly, the chip dimensions have been considerably miniaturized. For example, a typical 64K RAM chip is only 14.2 square millimeters, approximately the size of a printed letter [Ref. 4]. The drive towards larger capacity and decreased size is expected to continue into the 1990's.

This trend toward higher packing densities has in turn lead to a considerable increase in heat fluxes at both the chip and module levels. For reliable long term operation of the device, these large heat fluxes must be removed, while maintaining the components at acceptably low temperature

levels. To get an idea of the magnitude of the decreased size to increased heat flux relationship let us consider a power dissipation of 10 watts on a 5x5 mm chip. This yields a heat flux of 5 x 10⁵ w/m², which is only 20 times less than that on the surface of the sun. As seen in Fig. 1, the sun's surface temperature is more than 6000°C while today's chips must be designed to operate at considerably lower temperatures, between 100°C to 125°C [Refs. 5,6,7]. A strong need to keep the devices at these low temperature levels exists, since for every 20°C decrease in chip temperature, the chi failure rates are cut in half [Ref. 5].

B. IMMERSION COOLING: ANALYTICAL AND EXPERIMENTAL STUDIES

Immersion brings the liquid coolant into direct physical contact with the electronic package. It is therefore important that the coolant exhibit several crucial characteristics. It must be dielectric in nature so as not to adversely affect the circuits immersed in it. Also, it must be non-toxic as well as chemically inactive with the materials which compose the immersed portions of electronic package. Because very high heat transfer rates are attainable with direct immersion cooling, application has been receiving much attention in recent years [Refs. 3,8-10].

Having decided on the use of immersion cooling, the designer still has the choice of natural convection, forced convection or phase change as the cooling mechanism.

Because of its potentially high-dissipation capabilities, together with such added advantages as no noise and high reliability, natural convection is now generally recognized as an effective and viable means for proper thermal control of electronic packages [Ref. 11].

Two of the earliest studies in this area were conducted at Bell Telephone Laboratories by Baker [Refs. 12,13]. From his first study of free and forced convection cooling [Ref. 12], Baker showed that liquid immersion is an effective means of cooling small heat sources. The free convection cooling by liquid was found to be more than three times as effective as free convection cooling by air for the same device. For forced convection, the improvement was greater for liquid than air by a factor of 10.

In his second forced convection study using two different fluids [Ref. 13], boundary layer analysis showed that the convective heat transfer coefficient would increase significantly as the heat source size decreased. The convective heat transfer coefficient increased by a factor of 15 when the size was decreased from 2.00 to 0.01 square centimeters [Ref. 13]. A similar increase was noted for free convection.

More recently, Park and Bergles [Ref. 14] experimentally studied natural convection from discrete flush mounted and protruding heaters of varying widths, in water and R-113. They documented the increase in heat transfer coefficient

with decreasing width. Also, for protruding heaters, the heat transfer coefficients for the top heaters in an array were found to be higher than those for the bottom heaters. This trend was not observed for flush mounted heaters. As the distance between heaters increased, so did the heat transfer coefficients, the effect being greater in R-113 than in water.

Knock [Ref. 15] conducted a flow visualization study in a liquid filled rectangular enclosure with a single protruding heater from one vertical wall. Using water as the coolant, he found the existence of a dual-cell flow configuration where the upper cell was buoyancy driven and the lower cell was shear driven. He also concluded that as the heater height increased, the Nusselt number decreased.

Lin and Akins [Ref. 16] computed transient and steadystate natural convection heat transfer in a fluid-filled
cubical enclosure. The flow was initiated by subjecting
each of the six walls to a sudden change in temperature.
They concluded that the size of the enclosure determined the
circulation pattern.

Chu, et al. [Ref. 17] studied the natural convection in a rectangular enclosure with a long horizontal heat source mounted on one vertical wall. A number of different heater sizes and locations, as well as enclosure sizes, were investigated. They concluded that as the heater was moved downward on the wall, both the Rayleigh number and the

circulation rate increased. The heater size had little effect on circulation. A secondary flow cell was found to develop at the upper surface when the height of the enclosure was increased.

Yang, et al. [Ref. 11] used a three-dimensional finite difference method to study the natural convection cooling of a chip array in a rectangular enclosure filled with a dielectric liquid. They found the temperatures on the chip surfaces to be oscillatory, with wave forms ranging from simple to complex. The maximum chip surface temperatures occurred on the top row of chips for large gap sizes, but oscillated among all rows for small gas sizes.

It is clear from these studies that only a limited amount of information currently exists on natural convection in liquids from discrete protruding heat sources. The small size of the heat sources results in three-dimensional flow and temperature fields. The capability of numerically simulating these complex flows has only recently been developed for laminar flows, as an example in Yang et al. [Ref. 11]. Detailed measurements of transport are needed to verify such computations. Also, flow visualization studies are needed to examine the various possible flow regimes and the laminar to turbulent transition in such flows.

C. OBJECTIVES

The objectives of this study were:

- 1. to design and construct a liquid immersed vertical channel with discrete protruding heat sources. This geometry simulates a printed circuit board array with a number of heat dissipating electronic components. The heat sources were eight rectangular stainless steel protrusions, modeled geometrically after 20-pin dual-in-line packages (DIP). These were attached to the vertical card, in the form of a single vertical column.
- 2. to obtain the steady state natural convection flow pattern visualization within the interrupted channel, for a range of component power dissipation rates and plate spacings. The flow was visualized using a plane of light which illuminates suspended particles in the water.
- to measure component temperatures for various power inputs and card spacings and develop appropriate heat transfer correlations for this geometry.

This study was also intended to be a basis for future heat transfer experiments using various component array sizes and element spacings. The measurements will be used as a guide toward future experiments and computational efforts.

All objectives were achieved. As well a proving the reproducibility of both the numerical data and flow visualization photographs.

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II. EXPERIMENT

A. GENERAL DESIGN CONSIDERATIONS

The channel assembly, seen in Fig. 2, consisted of a vertical test surface with eight rectangular stainless steel blocks protruding from it mounted in a vertical column. A parallel shrouding plate was placed in front of the test surface. This configuration was meant to simulate one column of 20-pin DIP's mounted on a printed circuit board, with the back of another printed circuit board directly in front of it.

Each protrusion was heated by a foil heater mounted on the back (Fig. 3). The foil heater and the stainless steel block together act as a 20-pin DIP model and are hereafter referred to as a heater. A precision 2.0 \Omegat1\% resistor placed in series with each heater and its power supply, allowed the input power to be accurately determined. Five thermocouples, one on the center of each exposed face of the heater allowed for surface temperature measurements (Fig. 4). An additional thermocouple was mounted in the center of each heater mounting slot to measure the heater temperature.

The channel assembly was suspended in the center of a one cubic meter tank filled with purified water. Three thermocouples monitor the tank temperature at the top, middle and bottom. A computer-aided data acquisition system

was used to measure temperatures throughout the study. A line drawing of the entire system is seen in Fig. 3.

Several criteria were established to be used as guidelines for design and construction. The following is a list of these criteria and their implications.

1. Heater Block Dimensions

As previously noted, each heater was to geometrically model a 20-pin DIP. Micrometer measurements of an actual 20-pin DIP were made and each heater block was cut from solid 304 stainless steel to these dimensions (Figs. 6, 7). Stainless steel was used to prevent chemical reaction with the water of the immersion bath, and because it has a thermal conductivity in the same range as an actual DIP.

2. <u>Heating Element</u>

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Each block was to be heated individually and must be able to withstand temperatures in excess of 100°C, the boiling point of water. There was also the necessity to be able to accurately determine the temperature at the heater's surface, as well as having an even heating of the block.

During design, both an imbedded resistor and a cartridge type heater were considered, but the inability to accurately determine the temperature at their centers forced their elimination. The resistance type foil heater, however, met or exceeded all the requirements set for the

heating element and had the advantage of ease of installation.

3. <u>Heater Location</u>

The goal of this study was to model a single column of DIPs in a vertical channel. After examining several actual printed circuit cards, a configuration of eight protruding heaters in a single column was decided on. The heaters were spaced on one inch centers as found in many actual applications.

4. Visualization Technique

Visualization was accomplished with an eight milliwatt helium neon laser and a cylindrical lens (Fig. 8). The beam of light was split into a plane which illuminated particles suspended in the immersion bath water (Fig. 9). The particles were Pliolite, an inert pigment used in the manufacture of paints and adhesives. The particles have a specific gravity of 0.93, which results in a large suspension time in water.

This technique allowed for the visualization of a single two-dimensional plane of the flow field. Other planes can be examined by minor realignment of the laser-lens assembly. The method also has the benefit of allowing the bath to remain electrically nonconductive.

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5. Thermocouple Design and Placement

To accurately measure the temperature of each block face, individual thermocouples were employed. However, if

the thermocouple protrudes above the surface, it will affect the flow field. Therefore a groove was cut on each face to accommodate the thermocouple. If the thermocouples are placed significantly below the surface, they will not accurately reflect the surface temperature. To minimize this problem, 0.003 inch copper-constantan thermocouples were used, and they were placed in 0.02 inch radius grooves (Fig. 10). The larger groove allowed for the thickness of the bonding agent. The grooves were cut so that the bead of the thermocouple will be located at the center of each surface.

6. Other Considerations

The test surface was constructed of plexiglas to allow for easier milling of heater mounting slots. The back of the test surface was covered with foam rubber insulation to minimize conduction losses through the test surface. The outer surfaces of the immersion tank were also covered with foam rubber insulation to minimize heat transfer with the ambient air. Styrofoam blocks covered with teflon were floated on the surface of the bath to prevent contamination of water and minimize heat losses through the free surface.

B. COMPONENTS

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1. Heater

The heater assembly (Figs. 11, 12) consisted of a 0.94 inches (23.88 mm) long, 0.31 inches (7.87 mm) wide and 0.24 inches (6.10 mm) high 304 stainless steel block (Fig.

13). A resistance type foil heater was bonded to the back of each block using Omega Bond 101, a high thermal conductivity adhesive. The blocks have one groove cut into each side face with one end of the groove at the face center. A 0.04 inch diameter hole was drilled through the block from the front surface to allow for the passing of the thermocouple lead (Figs. 11, 12).

The foil heater consisted of a network of Inconel 600 conductor mounted on Kapton and is 1.37 inches (34.80 mm) long, 0.30 inches (7.62 mm) wide, and 0.007 inches (0.18 mm) thick (Fig. 14). Notches and holes were cut in the Kapton which align with the grooves and hole in the block. They allowed the passage of thermocouple wires, and also aided in the proper alignment of the foil heaters during bonding with the block. During the bonding of the foil heaters to the block, weight was applied to ensure uniform thickness of the adhesive and to also prevent curling at the edges. The power leads were soldered onto the protruding tabs of the foil heaters after the bonding agent had cured (Fig. 15).

The thermocouples were bonded in the grooves by placing a small drop of Eastman 910 adhesive on the thermocouple bead. The thermocouple was then placed in the designated groove. Using a straight pin, pressure was applied by hand until the adhesive set. After allowing for three hours of cure time, the remainder of the groove was

filled with Omega Bond 101 and smoothed to the level of the block face with a straight edge. The Omega Bond 101 experienced little shrinkage after curing. This procedure was repeated for each thermocouple.

2. Test Surface and the Shrouding Wall

Both the test surface and the shrouding wall were constructed of 1/2 inch (12.70 mm) thick plexiglas cut into a 12.0 inch (304.80 mm) square. Eight 0.015 inch (0.38 mm) deep, 1.41 inch (35.81 mm) long and 0.31 inch (7.87 mm) wide mounting slots were cut into the test surface on one inch (25.40 mm) vertical centers. This allowed the foil heaters to be mounted flush with the test surface while only the block protruded. Four 0.06 inch (1.524 mm) diameter holes were drilled through the test surface to conform to the grooves in each block. This allowed the thermocouple lead wires to pass through the test surface. One 0.12 inch (3.048 mm) diameter hole was drilled for the power leads to pass through (Fig. 16).

3. Test Surface Back Containment

Since all the wiring penetrated through the test surface, and the immersion bath was water, it was necessary to have a waterproof containment for the wiring. This was accomplished by fabricating a five sided box onto which the test surface was mounted (Fig. 17). The box was 12 inches (304.80 mm) square by 2.25 inches (57.150 mm) deep and was constructed of 0.50 inch (12.70 mm) plexiglas. A thin

groove was cut around the edges of the open face for a large O-ring gasket. The test surface was screwed over the open face with 20 stainless steel screws and compressed the O-ring to form a watertight seal. A 2 inch (50.80 mm) diameter plexiglas snorkel was mounted into the back of the containment box and extended to an elevation above the surface of the immersion bath. The wiring runs from the back of the test surface up through the snorkel.

4. Test Surface Support

Support for the test surface and shrouding wall was provided by an H style bracket which spanned the width of the tank (Fig. 18). The cross members between the span supports hold adjustable hangers to which the test surface and shrouding wall were attached, forming the vertical channel. These adjustable hangers allow the width of the vertical channel to be varied. They also allow the vertical adjustment of both the test surface and the shrouding wall.

5. Immersion Tank

The immersion tank was constructed such that the interior dimensions render a one cubic meter volume. The walls of the tank were 0.75 inch (19.050 mm) glass set in an aluminum frame. The glass was sealed watertight with Dow Corning 732 RTV adhesive/sealant (RTV) (Fig. 19).

6. Immersion Bath Filtration and Purification

Tap water was used and to ensure its purity and a resistivity of at least 0.1 megohn-cm, a Barnstead cartridge

filtration/purification system was employed (Fig. 20). consisted of four stages. The first stage contained a colloid/organic purification cartridge, followed by a high capacity deionization cartridge. The third stage was a high purity deionization cartridge, while the final stage was a 0.45 micron and larger filter cartridge. A suction was drawn from the bottom center of the tank with a small magnetic pump. The water passed through the four stage filtration/purification system and was returned to the top of the tank. Proper filtration/purification approximately 10 hours and one change of the colloid/organic and deionization cartridge was required. The filter cartridge did not require changing during a single filling of the tank.

C. ASSEMBLY

In addition to the thermocouples on the heater block assemblies, measurements of the mounting face temperature were made by placing a single thermocouple at the center of each mounting slot. These were bonded using Eastman 910 adhesive. This was the first step in the test surface assembly process.

STATES PARTICION

Land Control

Secreta 27 Miller

Once the heater assemblies, described in Section B, were ready, they were mounted on the test surface. For each heater block assembly the thermocouple wires and power leads were passed through their respective holes in the test surface (Fig. 21). A layer of Eastman 910 adhesive was then

applied to the bottom of the slot. Next, the heater assembly was firmly pressed into the mounting slot. This procedure was repeated for each heater assembly (Fig. 22).

Ten pounds of weight was distributed over all the protruding heaters and remains in place until the adhesive fully cured. Following the curing, the test surface was turned upside down and each lead was firmly cemented in this penetrating hole. The thermocouple and power leads were then assigned locations and routed through the snorkel. Next, the test surface was secured in place over the opening in the back containment box. The screw heads were covered with RTV and smoothed with a straightedge to the level of the test surface. The seam between the containment and test surface was also sealed with RTV.

Omega Bond 100 was used to fill in above the foil heater tab and in the seam around each heater assembly. This provided a watertight seal around each heater assembly and the test surface and also ensured a smooth and flat test surface (Fig. 23). Male plug-in connectors were then attached to each thermocouple lead wire and banana plug connectors to the power leads.

D. INSTRUMENTATION

Power to the Heaters

Each heater was run in series with a precision resistor that was measured to have a 2.02 ohm resistance. All eight heaters were in parallel with a Hewlett Packard

model number 6200B, 0-10 volts, 0-10 amperes, power supply. Both the source voltage and the heater voltage were measured independently. The current to each heater was then calculated by subtracting the heater voltage from the source voltage and dividing by the resistance of the precision resistor. The power input to each heater was calculated by the product of the heater voltage and heater current. Both voltages were measured by a Hewlett Packard model 3852S data acquisition system containing a Hewlett Packard model 44701A integrating vollmeter, all controlled by a Hewlett Packard model 300 computer.

2. <u>Temperature Measurement</u>

The thermocouples described earlier in Sections A.5 and C were referenced individually to an ice bath, as seen in Fig. 24. Each reference thermocouple was connected such that its constantan lead was connected to the constantan lead of the measurement thermocouple. The copper leads of each measurement and reference thermocouple were connected directly to a Hewlett Packard model 44705A relay multiplexer and inserted into the data acquisition system. The data acquisition system then measures the ice referenced voltage of each thermocouple. The voltages were converted directly into temperatures in the controlling computer program by using a fourth order polynomial, fit to the thermocouple manufacturer's calibration data for copper-constantan

thermocouples over a 10°C to 70°C range, with a maximum curve fit uncertainty of 0.00663°C RMS.

III. EXPERIMENTAL PROCEDURE

A. APPARATUS PREPARATION

A stainless steel, propeller type, variable speed stirrer was used to stir the immersion bath. Stirring was performed to remove temperature stratification and to disperse the particles for flow visualization. While stirring, the floating styrofoam blocks were removed and the test surface and shrouding wall, mounted on the support bracket, were lowered into the immersion bath and positioned near the center of the tank. The styrofoam blocks were then replaced on the surface of the immersion bath. Stirring continued for 5 to 10 minutes.

The laser was energized next. The cylindrical lens was placed in the beam path and rotated to obtain the plane of light of greatest intensity normal to the test surface. The laser was positioned to allow the plane of light to fall directly on the center of each heater. The light plane passed through both the front wall of the tank, through a 0.125 inch by 12.0 inch slit, and the shrouding wall. The slit was used to obtain a well-defined plane of light and also to prevent extraneous scatter light from entering the tank. A 35 mm camera and tripod were set up to visualize the flow through the right hand wall of the tank. Two, 2 inch wide strips of thin cardboard were positioned to allow the camera to only photograph the space between the test

surface and shrouding wall. A data back on the camera allowed for easy sequencing of the photographs for later analysis.

Crushed ice was placed into a stainless steel Dewar flask along with the ice reference thermocouples. A mercury thermometer inserted into the ice base ensured that the ice bath was at 0±0.05°C. The data acquisition system and the computer was then turned on. The internal voltmeter in the data acquisition system requires a minimum one hour warm-up period. During this hour, no current was passed through the heaters. The tank was allowed to achieve quiescence. After one hour passed, a visual inspection of the illuminated particles in the tank was performed to ensure that the immersion bath was quiescent.

The temperature acquisition program, contained in Appendix D, was loaded and the temperatures of all heater surfaces and the immersion bath was measured. When the results showed all surfaces to be within 0.10°C of each other, and the tank temperature stratification to be less than 0.10°C, the experiment was ready to begin. A chance exists that the tank may need to be restirred, and allowed to sit to quiescence for approximately one hour.

B. TEST PROCEDURE

1. <u>Initial Instrument Settings</u>

Once the immersion bath was quiescent and the temperature of the heaters and the bath were within the

required values, a sampling of all thermocouple temperatures was performed and printed. This record was saved and labeled as the baseline for the run. The power calculation program, contained in Appendix D, was loaded, and the power supply was set to zero and energized.

The power was increased incrementally and the output from the power calculation program was checked. This was performed repeatedly until the desired power setting was achieved. For this study, four power settings, 0.2, 0.5, 1.0 and 2.0 watts, were used for each channel width.

2. <u>Instrument Readings</u>

Thermocouple temperature measurements were taken every 10 minutes until successive temperature measurements remained unchanged to within 0.10°C. At this point, it was assumed that steady state had been achieved. This process took between one and two hours, depending on the channel width and power setting.

Once steady state was attained, the thermocouples were monitored once more and the temperatures and voltages recorded. Single thermocouples on various heater assemblies were also sampled over a period of several minutes to detect any temperature oscillations with time.

The total voltage drop across the power supply and across each heater was measured. The power for each heater was then calculated and printed by the computer program.

After all the data had been recorded, and with the power

supply and laser on, flow visualization photographs were taken.

3. Photographic Technique

A Nikon F3 series camera with a 50 mm f2.8 lens, a MF-18 data back, a MD-4 motor drive, and a MT-2 intervelometer was used for the photography. The film used was Kodak ASA400 Tri-X Pan black and white print film. The first picture taken was a blank and the data back was set to display the date. The laboratory lighting was turned off. Using a flashlight the camera was set to f2.8, and the focus adjusted. The intervelometer was set for the required exposure length. Four photographs of each channel width and power setting were taken as shown in Table 1.

TABLE I
PHOTOGRAPH EXPOSURE VARIATIONS

Picture #	<u>_f</u> _	Exposure Time
1	2.8	20 sec.
2	2.8	30 sec.
3	4.0	20 sec.
4	4.0	30 sec.

4. Experiment Completion

Once the photographs were taken, the laboratory lighting was turned on and the laser and power supply were

turned off. The propeller stirrer was started and the water immersion bath was mixed for 5 to 10 minutes. The immersion bath was now left to become quiescent again. After quiescence was achieved, the apparatus was ready to start another run with the same channel width at a different power level.

5. Channel Width

The channel widths, also called spacings, for this study were as follows;

- a. no shrouding wall
- b. 2.91 inches (73.810 mm)
- c. 0.47 inches (11.913 mm).

These spacings were measured from the test surface to the shrouding wall. Each heating element protrudes 0.22 inches (5.598 mm) into this spacing, in front of the test surface. The spacings were changed by hoisting the test surface and shrouding wall support bracket from the immersion bath and removing the installed spacer. The spacer corresponding to the next spacing desired was installed. The apparatus was then ready for a new set of tests.

D. DATA ANALYSIS

There is more than one vertical dimension involved in this study. Any one of these could be chosen as the characteristic length, L, for determining the nondimensional temperature, T, and the Modified Grashof number, Gr*. The 0.31 inch (7.874 mm) vertical height of the heater assembly

was chosen since it characterizes the local region of the buoyant flow. The corresponding temperature scale was easily obtained by combining the convected energy from each component with the component height and fluid thermal conductivity. An alternative would be the local downstream distance from the channel bottom. However, the appropriate temperature scale for this choice was not clear. The properties of the water were evaluated at the ambient bath temperature, T_{INF}. These values, listed in Table II, were obtained by linearly interpolating a table of properties [Ref. 18].

In order to determine the net convected energy from each heater, the conduction losses, Q_{COND}, were required to be determined for each power input and channel width. These were calculated from a resistance network consisting of the foam rubber insulation and the plexiglas. The net conduction loss is given by

$$Q_{COND} = \frac{\Delta T}{R_A + R_B} = \frac{T_H - T_{INF}}{R_A + R_B}$$
 (1a)

with

$$R_{\mathbf{A}} = \frac{\Delta X_{\text{PG}}}{k_{\text{PG}} A}$$

$$R_{\mathbf{B}} = \frac{\Delta X_{\text{R}}}{k_{\text{R}} A}$$
(1b)

TABLE II

PROPERTIES OF WATER [REF. 18]

Kinematic Viscosity X·10 ⁶ M ² /Sec	1.0453	1.0460	1.1030	1.0998	1.0992	1.0983	1.0981	1.0979	1.0970	1.0970	1.0970	1.0966
Expansion Coefficient $\beta \cdot 104$ 1/°C	1.899	1.897	1.648	1.662	1.664	1.668	1.669	1.670	1.674	1.674	1.674	1.675
Thermal Conductivity kf.103 w/m.°C	86.009	600.35	596.80	96.965	596.99	597.04	597.06	597.07	599.11	597.11	597.11	597.14
Spacing	73.81	73.81	73.81	73.81	11.913	11.913	11.913	11.913	No wall	No wall	No wall	No wall
O _{IN}	0.2	0.5	1.0	2.0	0.2	0.5	1.0	2.0	0.2	0.5	1.0	2.0
T _{INF}	18.43	18.32	16.08	16.20	16.22	16.25	16.26	16.27	16.30	16.30	16.30	16.31
Run	ı	7	m	4	2	9	7	ω	6	10	11	12

where A is the area normal to the direction of heat flow, and X_{PG} and X_{R} the thicknesses of plexiglas and foam rubber. See Table III.

TABLE III
PHYSICAL CONSTANTS

ITEM	VALUE	
A	0.000188	m ²
ΔX_{PG}	0.006731	m
Δx_R	0.003175	m

The thermal conductivity for each material was determined from a table of properties in Reference 19, and listed in Table IV.

TABLE IV
THERMAL CONDUCTIVITY OF MATERIALS

<u>Material</u>	$k \left(w/m - {}^{\circ}C \right)$
Plexiglas	0.1421
rubber insulation	0.0389

The temperature difference, ΔT , was assumed to be the difference between the heater, $T_{\rm H}$, and the ambient temperature, $T_{\rm INF}$. The convective thermal resistance on the outside of the insulation was neglected here. This

calculation of $Q_{\rm COND}$, therefore, was a "worst case" estimate, using a one-dimensional model. Maximum estimated conduction losses in this study were only about 1.5% of the energy input.

After calculating $Q_{\rm COND}$ and knowing the energy into the system, $Q_{\rm IN}$, a simple energy balance was used to determine the energy convected into the fluid, $Q_{\rm CONV}$:

$$Q_{CONV} = Q_{IN} - Q_{COND}$$
 (2a)

where

$$Q_{IN} = (\frac{V_T - V_H}{R}) V_H$$
 (2b)

and:

 V_T = input voltage

V_H = heater voltage

R = 2.02 ohms

The nondimensional temperature was next obtained as:

$$T = \frac{(T_{AVG} - T_{INF}) A k_f}{Q_{CONV} L}$$
 (3)

where T_{AVG} is the average of the temperatures measured on the five exposed surfaces of the heater assembly and L is the characteristic length.

The Modified Grashof number is defined as:

$$Gr* = \frac{g \beta L^4 Q_{CONV}}{A k_f v^2}$$
 (4)

A complete set of sample calculations is contained in Appendix A. Uncertainty calculations in the evaluation of the nondimensional parameters is presented in Appendix B.

IV. RESULTS

A. FLOW VISUALIZATIONS

Photographs of the natural convection flow in a plane passing through the geometric center of each component are presented in Figs. 25-27. Fig. 25 depicts flows with no shrouding wall in place. Figures 26 and 27 show flows with the shrouding wall 73.81 mm and 11.193 mm, respectively, in front of the test surface. Finally, Fig. 28 depicts one component power input without the shrouding wall in which the camera and laser positions were interchanged.

Observations for all the no wall case (Fig. 25), show the presence of a dual flow structure. Near the protruding heaters, the flow resembles flow past an obstruction. It is clearly visible that the flow follows the contour of the protrusions dipping nearer the test surface after passing over a block and rising before reaching the next block. Further away from the test surface a buoyant boundary layer structure is visible, as expected.

At the lowest power input setting, 0.2 watts, distinct particle traces are visible throughout the flow, indicating a strong two-dimensional behavior near the center of the block. As the power input is increased, these traces get shorter. This indicates more pronounced out-of-plane motion and hence a stronger three-dimensional flow.

It is also quite evident that as the power input is increased, the entrainment velocities also increase but the thickness of the buoyant layer remains vertically unchanged. This may again be due to larger three-dimensional effects.

It is of some interest to note that Fig. 25 shows no dead regions or vortices. Also, for these power levels, the particle traces indicate laminar flow. The effective origin for the outer boundary region flow is approximately one and a half heater spacings upstream of the lowest heater at the lowest power setting. It moves further upstream with increasing input power.

Observations with the shrouding wall 73.81 mm in front of the test surface (Fig. 26), are similar to the no wall case in many respects. However, the entrainment velocity is smaller, therefore the buoyant layer thickness is decreased. Also, the effective origin of the flow has moved to approximately one heater spacing upstream at the low power inputs. It is also apparent that for the 0.2 watt input power setting, quiescence was not achieved, prior to the start of the run.

For the closest spacing (Fig. 27) the flow still follows the contour of the test surface and there are no dead zones or eddy flow visible. Since the shrouding wall is so close to the test surface, the entire gap region participates in the transport, unlike for the two previous spacings, where the shrouding was either absent or was significantly far

from the test surface. It is interesting to note that at this spacing, the shrouding was receiving thermal energy from within the channel. This caused a boundary layer flow to develop on the back of the shrouding wall.

Observation of the flow in Fig. 28 reveals the presence of entrainment from the left and right sides of the heater assemblies.

B. QUANTITATIVE

Graphs of block number vs. excess temperature are contained in Figs. 29-46. Note block 1 is the upper most block in the channel. It is also noted that thermocouple #1 corresponding to block 1 heater and thermocouple #31 corresponding to block 6 front face were broken and their data_were not plotted. These figures allow a visual interpretation of the temperature across each block face at each spacing and power input. From these graphs and the data contained in Appendix C, no dramatic increase in temperature is apparent as the channel width is decreased.

Figure 47 is a comparison of the front surface temperature of the eight blocks at two watts with the self similar solution for a vertical uniform flux surface. The area of the flat plate is that formed by the eight blocks and the spacings between them. The uniform flux is then obtained by dividing the total convected energy with this area. The equation for the temperature excess at the surface is

$$(T_{SURF} - T_{INF}) = \left[\frac{Q_{CONV}}{1.172 k_{f}A}\right]^{4/5} \left[\frac{4v^{2}x}{g\beta}\right]^{1/5}$$
 (5)

where the constant 1.172 has been evaluated at Prandtl number equal to 6.7 for water. The fluid property values used were those of the 2.0 watt run with no wall, run 12. From Fig. 47, near the bottom of the heated protrusion column, the measured excess temperatures agree well with the similarity prediction. However, as the flow progresses up the channel, the actual data and the theoretical prediction diverge. This may be due to the increased three-dimensionality of the flow leading to more cooling of the heated blocks.

Results of the data analysis are contained in Appendix E in tabular form. These are also plotted in Figs. 48-50 as the modified Grashof number versus a nondimensional temperature T. We note that T is the inverse of the Nusselt number. Each graph is for a single channel width.

In all the graphs, a general trend is that as the modified Grashof number increases, the nondimensional temperature decreases, indicating higher Nusselt numbers. It is also apparent that the data for various blocks show less of a variation with increasing modified Grashof numbers. This indicates that the temperature variation for the different blocks is not directly proportional to the change in component energy dissipation. The data have been plotted over approximately a ten-fold increase in Q_{CONV}.

However, the resulting component temperature increases are only by a factor of 3 to 4. This is evident for all channel widths.

Another important is trend observed from the nondimensional temperature variations in Figs. 48-50. For the lowest power input, resulting in the smallest modified Grashof number, the nondimensional temperatures decrease as the shrouding plate is moved further away from the test surface. At higher values of modified Grashof number, no significant difference is observed between the different channel widths. The difference at lower modified Grashof numbers presumably results from the greater entrainment as the channel width is increased. For larger modified Grashof numbers, conduction to the shrouding wall may become appreciable, making the temperature differences between the various spacings less significant.

V. CONCLUSIONS

The author has found no previous natural convection liquid immersion cooling studies of a simulated column of protruding electronic devices with or without a shrouding wall. A direct comparison with other studies has therefore not been possible.

Flow visualization provided the evidence of a dual flow structure. Near the test surface, the protrusions govern the flow structure, while further away from the test surface, the flow is similar to a natural convection boundary layer. There were no dead zones or vortices observed for the conditions examined. As the input power increased, the three-dimensional effects become more predominant and the effective origin moved further upstream. Entrainment velocities increased with increasing power input.

The surface temperatures increased with increasing power but no dramatic trend in temperature from spacing to spacing was apparent. The component temperatures near the bottom of the channel agreed with that of a flat plate with constant heat flux. Further downstream, the measured temperatures were below the uniform flux surface prediction. Nondimensional temperatures for each block decreased as the modified Grashof number increased.

VI. RECOMMENDATIONS

While performing the experimental runs a number of possible improvements to the apparatus were noted which would enable better flow visualization. Review of the obtained data showed that using the same configuration the data set should be enlarged to allow for better correlation of channel spacing and input power. These recommendations are stated below.

A. IMPROVEMENT TO EXPERIMENT

Apparatus

WARREN WARREN

Both the test surface and the shrouding wall should be _painted a dark, flat color, with only a thin slit remaining unpainted on the shrouding wall to allow passage of the plane of light.

A metal plate with a 12 inch long slit should be manufactured. It should have two leveling screws and hang from the top of the tank. This would allow for easier alignment of the plane of light with the test surface.

A similar device with an adjustable width slit should be manufactured to aid with the photographing of the flow.

The laboratory should be made lightproof, thus removing the need for experimentation at night only.

2. <u>Data Acquisition</u>

Data acquisition programs should be rewritten to include lines for the storage of acquired data. Also, the plotter should be interfaced with the system so that results could be directly plotted.

B. ADDITIONAL EXPERIMENTAL WORK

It is suggested that the following areas of study be experimentally explored:

- 1. Using the same test surface, several channel widths between 73.81 mm and 11.913 mm should be investigated to better understand the effects of card spacing. Note that at 11.913 mm thermal energy was conducted to the shrouding wall. It is recommended that the minimum channel spacing not be less than 11.913 mm, and that the minimum width where conduction to the shrouding wall is not present be found.
- 2. Using the same test surface, the input powers should be increased above 2.0 watts. Levels of 3.0, 4.0 and 5.0 watts are recommended. This will allow for a better power input to spacing correlation as well as seeing if the nondimensional temperature reaches a constant value.
- 3. Using the same test surface, only a selected number of blocks could be heated. This will allow an understanding of how the protrusion affects the flow.
- 4. A different test surface could be constructed and studied based on the above results. A 3 by 3 array of heater assemblies is recommended to begin with.

APPENDIX A

SAMPLE CALCULATIONS

A. DETERMINATION OF INPUT POWER

Using the data for block 2, run 1, Appendix C, the input power is calculated, using Equation (3b), to be:

$$Q_{IN} = \frac{(1.75 - 1.48) \cdot 1.48}{2.02} = 0.20 \text{ watts}$$

B. NONDIMENSIONAL TEMPERATURE

Using the same data as above, the heat loss via conduction through the test surface is calculated. Employing Equations (1b) and (1c), as well as the information in Tables II and III, and Appendix C, the resistances and area are calculated to be:

$$A = (0.0079)(0.0239) = 0.000188 \text{ m}^2$$

$$R_A = \frac{0.006731}{(0.1421)(0.000188)} = 251.96 \text{ °C/W}$$

$$R_B = \frac{0.003175}{(0.0389)(0.000188)} = 434.15 \text{ °C/W}$$

From Equation (1a), Q_{COND} is then calculated:

$$Q_{COND} = \frac{(20.63 - 18.32)}{251.96 + 434.15} = 0.003 \text{ watts}$$

From Equation (2a), Q_{CONV} is:

$$Q_{CONV} = 0.20 - 0.003 = 0.197$$
 watts

The average temperature of the convecting faces is:

$$T_{AVG} = \frac{(19.86 + 19.76 + 19.74 + 19.96 + 19.92)}{5.0} = 19.85$$
°C

The ambient temperature, $T_{\mbox{\footnotesize{INF}}}$, is taken as the average at three tank temperatures.

$$T_{INF} = \frac{(18.36 + 18.33 + 18.78)}{3.0} = 18.32$$
°C

From Equation (3) and Table II, the nondimensional temperature is found to be:

$$T = \frac{(19.85 - 18.32)(0.000188)(0.60038)}{(0.197)(0.007874)} = 0.11$$

C. MODIFIED GRASHOF NUMBER

Using Table II and Equation (4), the Modified Grashof number is calculated to be:

$$Gr* = \frac{(9.81)(1.897 \times 10^{-4})(0.197)(0.007874)^{4}}{(0.000188)(0.60035)(1.0460 \times 10^{-6})^{2}}$$

$$Gr* = 1.14 \times 10^5$$

APPENDIX B

UNCERTAINTY ANALYSIS

TABLE V

UNCERTAINTY VARIABLES

<u>Variable</u>	Uncertainty	<u>Basis</u>
Voltmeter Resolution	0.026°C 1.0 μV	Manufacturer data
Ice Bath Temperature	0.05°C	Manufacturer Calibration data
Polynomial Tempera- ture Conversion	0.00663 ^O C RMS	Polynomial fit error calculation
R	1.0%	Manufacturer data
L	0.0000254 M	Resolution of measurement device
kf	0.008 W/m·°C	[Ref. 18]
k _{pg}	5.0%	[Ref. 19]
k _r	7.0%	[Ref. 19]
β	0.0000535 1/°C	[Ref. 18]
ν	$0.00012 \text{ m}^2/\text{sec}$	[Ref. 18]
$\delta T_{H} = [(\delta VR)^{2} + (\delta I.$	B.) 2] $^{1/2} + \delta$ curve	
$\delta T_{\rm H} = [(0.025)^2 + (0.025)^2]$	$(0.05)^2 1^{1/2} + 0.00663$	

$$\delta T_{\rm H} = 0.063$$

$$\frac{\delta Q_{\text{IN}}}{Q_{\text{IN}}} = \left[\left(\frac{\delta R}{R} \right)^2 + \left(\frac{\delta V_{\text{H}}}{V_{\text{H}}} \right)^2 + \left(\frac{\delta V_{\text{T}}}{V_{\text{T}}} \right)^2 \right]^{1/2}$$

$$= \left[\left(\frac{0.02}{2.02} \right)^2 + \left(\frac{1 \times 10^{-6}}{2.32} \right)^2 + \left(\frac{1 \times 10^{-6}}{2.75} \right)^2 \right]^{1/2}$$

$$= 0.010$$

$$\frac{\delta A}{A} = \left[\left(\frac{0.0000254}{0.007874} \right)^2 + \left(\frac{0.0000254}{0.023876} \right)^2 \right]^{1/2}$$

$$= 0.0034$$

= 0.07

$$\frac{\delta R_{A}}{R_{A}} = \left[\left(\frac{\delta X_{pg}}{X_{pg}} \right)^{2} + \left(\frac{\delta A}{A} \right)^{2} + \left(\frac{\delta k_{pg}}{k_{pg}} \right)^{2} \right]^{1/2}$$

$$= \left[\left(\frac{0.0000254}{0.006731} \right)^{2} + \left(0.0034 \right)^{2} + \left(0.05 \right)^{2} \right]^{1/2}$$

$$= 0.050$$

$$\frac{\delta R_{B}}{R_{B}} = \left[\left(\frac{\delta X_{r}}{X_{r}} \right)^{2} + \left(\frac{\delta A}{A} \right)^{2} + \left(\frac{\delta k_{r}}{k_{r}} \right)^{2} \right]^{1/2}$$

$$= \left[\left(\frac{0.0000254}{0.003175} \right)^{2} + \left(0.0034 \right)^{2} + \left(0.07 \right)^{2} \right]^{1/2}$$

$$\frac{Q_{\text{COND}}}{Q_{\text{COND}}} = \left[\left(\frac{\delta R_{\text{A}}}{R_{\text{A}}} \right)^{2} + \left(\frac{\delta R_{\text{B}}}{R_{\text{B}}} \right)^{2} + \left(\frac{\delta T_{\text{H}}}{T_{\text{H}}} \right)^{2} + \left(\frac{\delta T_{\text{INF}}}{T_{\text{inf}}} \right)^{2} \right]^{1/2}$$

$$= \left[(0.05)^{2} + (0.07)^{2} + \left(\frac{0.063}{20.63} \right)^{2} + \left(\frac{0.063}{18.32} \right)^{2} \right]^{1/2}$$

$$= 0.0861$$

$$\frac{\delta Q_{\text{CONV}}}{Q_{\text{CONV}}} = \left[\left(\frac{\delta Q_{\text{COND}}}{Q_{\text{COND}}} \right)^2 + \left(\frac{\delta Q_{\text{IN}}}{Q_{\text{IN}}} \right)^2 \right]^{1/2}$$
$$= \left[(0.086)^2 + (0.01)^2 \right]^{1/2}$$

$$= 0.087$$

$$\frac{\delta T}{T} = \left[\left(\frac{\delta A}{A} \right)^2 + \left(\frac{\delta k_f}{k_f} \right)^2 + \left(\frac{\delta T_{AVG}}{T_{AVG}} \right)^2 + \left(\frac{\delta T_{inf}}{T_{inf}} \right)^2 + \left[\frac{\delta Q_{CONV}}{Q_{CONV}} \right] + \left(\frac{\delta L}{L} \right)^2 \right]^{1/2}$$

$$= \left[\left(0.0034 \right)^2 + \left(\frac{0.0038}{0.60038} \right)^2 + \left(\frac{0.063}{19.85} \right)^2 + \left(\frac{0.063}{18.32} \right)^2 + \left(0.087 \right)^2 + \left(\frac{0.0000254}{0.007874} \right)^2 \right]^{1/2}$$

$$= 0.0835$$

$$\frac{Gr^*}{Gr^*} = \left[\left(\frac{\delta B}{B} \right)^2 + 4 \left(\frac{\delta L}{L} \right)^2 + \left(\frac{\delta Q_{CONV}}{Q_{CONV}} \right)^2 + \left(\frac{\delta A}{A} \right)^2 + \left(\frac{\delta k_f}{k_f} \right)^2 + 2 \left(\frac{\delta V}{V} \right)^2 \right]^{1/2}$$

$$= \left[\left(\frac{53.5 \times 10^{-6}}{0.1897 \times 10^{-3}} \right)^2 + 4 \left(\frac{0.0000254}{0.007874} \right)^2 + (0.087)^2 - (0.0034)^2 + \left(\frac{0.008}{0.60038} \right)^2 + 2 \left(\frac{0.00012}{0.1046 \times 10^{-5}} \right)^2 \right]^{1/2}$$

$$= 162.24$$

The above uncertainty calculations are intended to be a representative example of the overall uncertainty for this study.

APPENDIX C

TABULAR DATA

Run	# 7
RULI	# 1

Spacing: 73.81 mm

Power: 0.2 watts

Date: 11 November 1987 BLOCK #!

T. C. T. C. T. C. T. C. T. C.	# 4 # 5	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00079035 .00077911 .00077978 .00079212 .00079029	Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C	20.0243445765 :9.7450558912 :9.7617177375 20.0663090423 20.0228541866 253317.564556
T. C. T. C. T. C. T. C. T. C.	# 9 # 10 # 1!	Volts D.C.	.00078393 .00077967 .00077895 .00078759 .00078618 .00081496	Temp. DEG. C	19.8648471972 19.759983891 19.7410892474 19.9557819586 19.9207517085 20.8352713352
T. C. T. C. T. C. T. C. T. C. T. C.	# 14 # 15 # 16 # 17	BLOCK #3 Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00078062 .00077524 .00077458 .00078948 .00078817	Temp. DEG. C	19.7825938819 19.6488717936 19.6324647191 20.0027334804 19.9701908396 20.8599881386
T. C. T. C. T. C. T. C. T. C. T. C.		BLOCK #4 Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00078141 .00077793 .00077313 .00078624 .00078245	Temp. DEG. C	18.9022265892 19.7132518396 19.5964169503 19.9222424083 19.8280709822 20.8386618334

BLOCK #5

T. C. T. C. T. C. T. C. T. C.	# 25 # 26 # 27 # 28 # 29 # 30	Volts D.C.	.00077637 .00077434 .00077308 .00073541 .00077941	Temp. Temp. Temp. Temp.	DEG. DEG. DEG. DEG. DEG.	0000	19.6769614222 19.6264983744 19.5951738766 19.9016206587 19.7525220116 20.5782076142
		BLOCK #6					
T. C. T. C. T. C. T. C. T. C.	# 31 # 32 # 33 # 34 # 35 # 36	Volts D.C.	.0271272 .00077167 .00077268 .00078155 .00077997	Temp. Temp. Temp.	DEG. (CDEG. (C)))))))))))))))))))))	0000	4348.10146544 19.5601179051 19.5852291742 19.8057057213 19.7664398002 20.5279853066

BLOCK #7

T. C. T. C. T. C. T. C. T. C.	# 38 # 39 # 40 # 4!	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00076535 .00076!1 .00076053 .00077596 .00077246	Temp. DEG. C	19.297242815 19.293062966 19.5667597958
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BLOCK #8

T. T. T.	c. c. c. c.	# # #	44 45 46 47	Volts Volts Volts Volts	D.C. D.C. D.C.	.0007653 .00075635 .00075687 .00077204 .00076312	Temp. Temp. Temp. Temp.	DEG. DEG. DEG. DEG.	0000	19.4017132152 19.1790649214 19.192003674 19.5693172318 19.3474909198
Τ.	C.	#	48	Volts	D.C.	.00079523				20.1455478033

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.00072357	TEMP. DEG. C	18.362739297
VOLTS D.C.	.00072241	TEMP. DEG. C	18.3338268635
VOLTS D.C.	.00072041	TEMP. DEG. C	18.2839738639

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BLOCK # 1 HEATER VLOTS D.C. 1.474186
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BLOCK # 2 HEATER VLOTS D.C. 1.482119

BLOCK # 3 HEATER VLOTS D.C. 1.475075

BLOCK # 4 HEATER VLOTS D.C. 1.475035

BLOCK # 5 HEATER VLOTS D.C. 1.47361

BLOCK # 6 HEATER ULOTS D.C. 1.47488

BLOCK # T HEATER VLOTS D.C. 1.475371

BLOCK # 8 HEATER VLOTS D.C. 1.478689

INPUT VLOTAGE D.D. VLOTS 1.756932

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BLOCK # 1 HEATER POWER WATTS .205345631067
BLOCK # 2 HEATER POWER WATTS .201636420172
BLOCK # 3 HEATER POWER WATTS .205821288255
BLOCK # 4 HEATER POWER WATTS .205845515542
BLOCK # 5 HEATER POWER WATTS .205686204168
BLOCK # 6 HEATER POWER WATTS .205937056317
BLOCK # 7 HEATER POWER WATTS .205646997095
BLOCK # 8 HEATER POWER WATTS .203680625459
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Run #2

CONTROL TO SERVED THE CONTROL OF THE

T. C.

24

Spacing: 73.81 mm

0.5 watts Power:

11 November 1987 Date:

BLOCK #1

Volts D.C.

		5230				
T. C. T. C. T. C. T. C. T. C.	# 3 # 2 # 2	Volts D.C.	.00084358 .00084743 .00087354 .00085713	Temp. DE Temp. DE Temp. DE Temp. DE	6. C	21.7959413132 21.3447868234 21.4401533038 22.0854206622 21.9278413532 -204.164485618
		BLOCK #2				
T. G. T. G. T. G. T. G. T. G.	# 8 # 9 # 10 # 11	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00035608 .0002405! .00084449 .00086327 .00086!59	Temp. DE Temp. DE Temp. DE Temp. DE	6. C	21.5543504051 21.2687280112 21.3673296705 21.5471712547 21.7907437714 23.5119106095
		BF00K #3				
T. C. T. C. T. C. T. C. T. C.	# 14 # 15 # 16 # 17	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00084644 .0008356 .00083746 .0008556 .00086472 .00085167	Temp. DE Temp. DE Temp. DE Temp. DE	16. C 16. C	21.465168443 21.1470588682 21.1931529803 21.8699825115 21.8682061615 24.0151694159
		BLOCK #4				
T. C. T. C. T. C. T. C. T. C.	# 20 # 21 # 22 # 23	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00084549 .00093883 .00083459 .0008513 .00085424	Temp. DE	16. C	21.1220274501 21.7835661228 21.608794953

.00095168

Temp. DEG. C 24.015415791

BLOCK #5

Τ.	C.	# 25	Volts D.C.	.00084001	Temp. DEG. C	21.2563394622
Τ.	С.	# 26	Volts D.C.	.00083327	Temp. DEG. C	21.0893111906
Т.	С.	# 27	Volts D.C.	.00083464	Temp. DEG. C	21.1232666593
Τ.	С.	# 28	Volts D.C.	.00095408	Temp. DEG. C	21.8523682058
		# 29	Volts D.C.	.00085179	Temp. DEG. C	21.5481302839
Τ.	С.	# 30	Volts D.C.	.00093206	Temp. DEG. C	23.5317872731

BLOCK #6

Τ.	С.	#	31	Volts	D.C.	.0204947	Temp.	DEG.	C	1561.85597436
Τ.	С.	#	32	Volts	D.C.	.00082777	Temp.	DEG.	С	20.9529698935
Ή.	С.	#	33	Volts	D.C.	.00083079	Temp.	DEG.	С	21.0278383615
Τ.	С.	#	34	Volts	D.C.	.00085193	Temp.	DEG.	С	21.5515970394
Τ.	С.	#	35	Volts	o.c.	.00084676	Temp.	DEG.	С	21.4235583962
T.	С.	#	36	Volts	D.C.	.00092735	Temp.	DEG.	C	23.415615182

BLOCK #7

T. C.	# 37	Volts D.C.	.0009184	Temp.	DEG. C	20.7206064139
T. C.	# 38	Volts D.C.	.00080663	Temp.	DEG. C	20.4285698691
T. C.	# 39	Volts D.C.	.00081072	Temp.	SEG. C	20.5300704385
T. C.	_ #_ 40	Volts D.C.	.00083772	Temp.	DEG. C	21.1995958973
T. C.	# 41	Volts D.C.	.00083321	Temp.	DEG. C	21.0878240359
T. C.	# 42	Volts D.C.	.00091659	Temp.	DEG. C	23.1501157605

BLOCK #8

Τ.	С.	#	43	Volts	D.C.	.00081677	Temp.	DEG.	С	20.6801733318
Τ.	С.	#	44	Volts	D.C.	.00080155	Temp.	DEG.	С	20.3024714367
Τ.	С.	#	45	Volts	D.C.	.0008017	Temp.	DEG.	С	20.3061952802
Τ.	С.	#	46	Volts	D.C.	.00083331	Temp.	DEG.	С	21.0903026246
Т.	C.	#	47	Volts	D.C.	.00081589	Temp.	DEG.	С	20.6583430413
Τ.	С.	#	48	Volts	D.C.	.00089782	Temp.	DEG.	С	22.6866252613

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.00072415	TEMP. DEG.	C	18.3771948788
UOLTS D.C.	.00072244	TEMP. DEG.	С	18.3345746201
VOLTS D.C.	.00071894	TEMP. DEG.	С	18.2473286999

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BLOCK # 1 HEATER VLOTS D.C. 2.308692
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BLOCK # 2 HEATER VLOTS D.C. 2.321047
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BLOCK # 3 HEATER VLOTS D.C. 2.310025
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BLOCK # 4 HEATER VLOTS D.C. 2.309965
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BLOCK # 5 HEATER VLOTS D.C. 2.307129

BLOCK # 8 HEATER VLOTS 0.0. 2.309733

BLOCK # T HEATER VLCTS D.C. 2.310471

BLOCK # 8 HEATER VLOTS D.C. 2.315697

INPUT VLOTAGE D.C. VLOTS 2.75119

BLOCK # 1 HEATER POWER WATTS .505738412185 BLOCK # 2 HEATER POWER WATTS .494248574119 BLOCK # 3 HEATER POWER WATTS .50450602927 BLOCK # 4 HEATER POWER WATTS .504561538181 BLOCK # 5 HEATER POWER WATTS .5071811935 BLOCK # 6 HEATER POWER WATTS .5047761391 BLOCK # 7 HEATER POWER WATTS .504093301311 BLOCK # 8 HEATER POWER WATTS .499242491892

Run #3

T. C. # 23

T. C. # 24

Spacing: 73.81 mm

Power: 1.0 watt

Date: 16 November 1927

BLOCK #1

T. G. T. G. T. G. T. G. T. G. T. G.	# 4 # 5	Volts D.C.	.00085264 .0008407 .00084341 .00085203 .00085072 .0718988	Temp. [] Temp. [] Temp. [] Temp. [] Temp. [] Temp. []	DEG. C DEG. C DEG. C	21.340575407 22.2963785306 21.789210509
		BLOCK #2				
	# 8 <u>#</u> 9 # 10 # 11	Volts D.C.	.00084097 .00082733 .00083003 .00085922 .00085725 .00100509	Temp. [Temp. [Temp. [Temp. [Temp. [DEG. C DEG. C DEG. C	21.0089983693 21.7320819645
		BLOCK #3				
T. C. T. C. T. C.	# 15	Volts D.C.	.00084813 .00082149 .00082354 .00087966 .0008622 .00104456	Temp. [Temp. [Temp. [Temp. [Temp.]	DEG. C DEG. C DEG. C	21.4574906652 20.797246471 20.8480851991 22.2377774762 21.8058412397 26.2983644288
		BLOCK #4				
T. C. T. C. T. C.	-	Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.0008555 .00081905 .000824 .00087152	Temp. D Temp. D Temp. D	DEG. C	21.6399909929 20.7367291048 20.8594921586 22.0389264933

.00105802

Temp. DEG. C 21.7996538091 Temp. DEG. C 26.6283138115

Volts D.C. .00086195

Volts D.C.

BLOCK #5

т. (c.	#	25	Volts	D.C.	.00083707	Temp.	DEG.	С	21.1834884457
T. (C.	#	26	Volts	D.C.	.00082369	Temp.	DEG.	С	20.8518048988
T. (c.	#	27	Volts	D.C.	.0008239	Temp.	DEG.	С	20.8570124309
T. (c.	#	28	Volts	D.C.	.00087763	Temp.	DEG.	С	22.187577725
T. (C.	#	29	Volts	D.C.	.0008539	Temp.	DEG.	С	21.6003766327
T. (С.	#	30	Volts	D.C.	.00102435	Temp.	DEG.	С	25.8025269276

BLOCK #6

Τ.	С.	# 31	· Volts D.C.	.0268306	Temp. DEG. C	4169.36370749
Τ.	С.	# 32	Volts D.C.	.00080957	Temp. DEG. C	20.5015332825
Τ.	С.	# 33	Volts D.C.	.00081653	Temp. DEG. C	20.6742197127
Τ.	С.	# 34	Volts D.C.	.00085761	Temp. DEG. C	21.6922275192
Τ.	€.	# 35	Volts D.C.	.00084015	Temp. DEG. C	21.2598082976
Τ.	С.	# 36	Valts D.C.	.00102488	Temp. DEG. C	25.8:55385789

BLOCK #7

Τ.	C.	<u>#</u> 37	Volts D.C.	.00080817	Temp. DEG.	С	20.4667901543
T.	С.	# 38	velts D.C.	.0007829	Temp. DEG.	С	19.8392532308
τ.	С.	# 39	Volts D.C.	.00079324	Temp. DEG.	C	20.0961253311
Τ.	С.	# 40	Volts D.C.	.00084154	Temp. DEG.	ε	21.2942474331
Τ.	С.	# 41	Volts D.C.	.00082852	Temp. DEG.	С	20.9740433028
Τ.	С.	# 42	Volts D.C.	.00100595	Temp. DEG.	C	25.3506541914

BLOCK #8

T.	C.	# 43	Volts D.C.	.0008016	Temp. DEG.	С	20.303712721
Τ.	C.	# 44	Volts D.C.	.00077564	Temp. DEG.	С	19.6588152085
Τ.	С.	# 45	Volts D.C.	.0007749	Temp. DEG.	С	19.6404197327
Τ.	С.	# 46	Volts D.C.	.00083422	Temp. DEG.	С	21.1128572041
Τ.	C.	# 47	Volts D.C.	.00079941	Temp. DEG.	С	20.2493415233
T.	С.	# 4 8	Volts D.C.	.0009841	Temp. DEG.	С	24.8135071922

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.00063358	TEMP. DEG.	С	16.114752114
VOLTS D.C.	.00063229	TEMP. DEG.	С	16.082453277
VOLTS D.C.	.00063074	TEMP. DEG.	С	16.0436418264

BLOCK # 1 HEATER VLOTS D.C. 3.2881

BLOCK # 2 HEATER VLOTS D.C. 3.30615

BLOCK # 3 HEATER VLOTS D.C. 3.28981

BLOCK # 4 HEATER VLOTS D.C. 3.28965

BLOCK # 5 HEATER VLOTS D.C. 3.28679

BLOCK # 6 HEATER VLOTS D.C. 3.28936

BLOCK # 7 HEATER VLOTS D.C. 3.2906

BLOCK # 8 HEATER VLOTS D.C. 3.29782

INPUT VLOTAGE D.C. VLOTS 3.91787

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BLOCK # ! HEATER POWER WATTS !.02512214703
BLOCK # 2 HEATER POWER WATTS 1.00120696931
BLOCK # 3 HEATER POWER WATTS 1.02287033099
BLOCK # 4 HEATER POWER WATTS 1.02308115
BLOCK # 5 HEATER POWER WATTS 1.02546319485
BLOCK # 6 HEATER POWER WATTS 1.02346319485
BLOCK # 7 HEATER POWER WATTS 1.02182904059
BLOCK # 8 HEATER POWER WATTS 1.01228380743

Run #4

Spacing: 73.81 mm

Power: 2.0 watts

Date: 16 November 1987

BLOCK #1

T. 0. T. 0. T. 0. T. 0. T. 0.	# 3 # 4 # 5	Volts D.C.	.00099999 .00098964 .0009904 .00104:59 .00101351 0526:8:	Temp. [Temp. [Temp. [DE6. C DE6. C DE6. C	24.9634436235
		BLOCK #2				
T. C. T. C. T. C. T. C. T. C.	# 9 # 10 # 11	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00098101 .00095578 .0009585 .00102363 .00101402 .00131394	Temp. [Temp. [Temp. [Temp. [DEG. C DEG. C DEG. C	25.7846528787 25.5488916062
		BLOCK #3				
T. S. T. C. T. G. T. C. T. C.	# 14 # 15 # 16 # 17	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00096566 .00094913 .00093948 .00102837 .00098013 .0013708	Temp. [Temp. [Temp. [Temp. [DEG. C	23.7147451812 25.9011951721
		BLOCK #4				
T. C. T. C. T. C. T. C. T. C.	# 21 # 22 # 23	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00096349 .00093199 .00093953 .00098074 .00099323	Temp. [Temp. [Temp. [Temp. [DEG. C DEG. C DEG. C	23.5300609272 23.7159778176 24.7308541777 25.0380259348
1	# 24	Volts D.C.	.00137817	Temp. [リヒじ・ レ	34.4104326334

BLOCK #5

T. C. T. C. T. C. T. C. T. C. T. C.	. #	26 27 28 29	Volts D.C.	.00097024 .00095663 .00095773 .00104532 .0010185 .00135099	Temp. Temp. Temp. Temp.	DEG. DEG. DEG. DEG.	0000	24.4724726756 24.1373561325 24.1644498276 26.317000557 25.6589065951 33.7546109452
			BLOCK #6					

Τ,	C.	#	31	Volts	D.C.	0831284	Temp.	DEG.	С	405992.00471
T.	С.	#	32	Volts	D.C.	.00093437	Temp.	DEG.	C	23.58875325
T.	С.	#	33	Volts	D.C.	.00095337	Temp.	DEG.	С	24.0570513909
T.	С.	#	34	Volts	D.C.	.00101723	Temp.	DEG.	С	25.6277218494
Τ.	С.	#	35	Volts	D.C.	.00099699	Temp.	DEG.	С	25.1304590652
Τ.	C.	#	36	Volts	D.C.	.00136077	Temp.	DEG.	С	33.9906937561

BLOCK #7

Τ.	C.	#	37	Volts	D.C.	.00094894	Temp.	DEG.	C	23.9479043329
Τ.	С.	#	38	Volts	D.C.	.0009049	Temp.	DEG.	0	22.8615046201
Τ.	С.	#	39	Volts	D.C.	.00092636	Temp.	DEG.	C	23.3911933139
Ŧ.	C.	#	40	Volts	D.C.	.00101444	Temp.	DEG.	С	25.5592065735
T.	C.	#	41	Volts	D.C.	.00098306	Temp.	DEG.	С	24.7879256219
T.	С.	#	42	Volts	D.C.	.0013394	Temp.	DEG.	С	33.4746860386

BLOCK #8

Τ.	ε.	#	43	Volts	D.C.	.00092161	Temp.	DEG.	С	23.2740005238
Τ.	С.	#	44	Volts	D.C.	.00088689	Temp.	DEG.	С	22.4165257577
Τ.	€.	#	45	Volts	D.C.	.00088349	Temp.	DEG.	C	22.3324752532
T.	С.	#	46	Volts	D.C.	.00099579	Temp.	DEG.	С	25.1009610456
Τ.	С.	#	47	Volts	D.C.	.00092944	Temp.	DEG.	С	23.4671684316
T.	С.	#	48	Volts	D.C.	.00128954	Temp.	DEG.	С	32.2685944092

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.00063761	TEMP. DEG.	С	16.2156411894
VOLTS D.C.	.00063615	TEMP. DEG.	Ç	16.1790931662
VOLTS B.C.	.00063754	TEMP. DEG.	\mathcal{C}	16.2139889481

BLOCK # 1 HEATER VLOTS D.C. 4.62592

BLOCK # 2 HEATER VLOTS D.C. 4.65012

BLOCK # 3 HEATER VLOTS D.C. 4.62731

BLOCK # 4 HEATER VLOTS D.C. 4.62709

BLOCK # 5 HEATER VLOTS D.C. 4.62453

BLOCK # 6 HEATER VLOTS D.C. 4.52678

BLOCK # 7 HEATER VLOTS D.C. 4.62881

BLOCK # 8 HEATER VLOTS D.C. 4.63865

INPUT VLOTAGE D.C. VLOTS 5.50953

BLOCK # 1 HEATER POWER WATTS 2.02351939168 BLOCK #-2 HEATER POWER WATTS 1.97839585604 BLOCK # 3 HEATER POWER WATTS 2.02094328129 BLOCK # 4 HEATER POWER WATTS 2.02135113842 BLOCK # 5 HEATER POWER WATTS 2.02609358911 BLOCK # 5 HEATER POWER WATTS 2.02192576485 BLOCK # 7 HEATER POWER WATTS 2.01816116

BLOCK # 8 HEATER POWER WATTS 1.99985520396

Spacing: 11.913

Power: 0.2 watts

Date: 17 November 1987

T. C. T. C. T. C. T. C. T. C. T. C.	# 4 # 5	Volts D.C.	.00070914 .00070222 .00070959 .00071276 .00071114 .0651422	Temp. DEG. C	18.0029581147 17.8303297555 18.0141818604 18.093239593 18.0528394784 143771.001054
		BLOCK #2			
T. G. T. G. T. G. T. G.	# 8 # 9 # 10	Volts D.C.	.00070498 .0007008 .00069983 .00071178 .0007084 .00073574	Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C	17.8991887536 17.7948985648 17.7706941095 18.0688004516 17.9845007343 18.5659585444
		BLOCK #3			
	# 14 # 15 # 16 # 17	Volts D.C.	.00070403 .0005988 .00059779 .00071117 .00070759	Temp. DEG. C	17.9754893689 17.7449911752 17.7197860303 18.0535876605 17.9667909783 18.8208748884
		BLOCK #4			
	1 19 1 00 1 0 1 00 1 00 1 03 1 04	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Polts D.C. volts D.C.	.00071244 .00059555 .00069395 .00070627 .00070348	Temp. DEG. C	18.0852596654 17.6641305202 17.6214487367 17.931369563 17.9617665744 18.8425379729

T. C. T. C. T. C. T. C. T. C.	# 26 # 27 # 28 # 29	Volts D.C.	.00069733 .000693 .00069395 .00070453 .00070072	Temp. Temp. Temp. Temp.	DEG. CODEG. CODEG. CO	17.7083060338 17.6002312761 17.6239448487 17.8879623972 17.7929023659 18.6004549184
		BLOCK #6				
T. C. T. C. T. C. T. C. T. C.	# 32 # 33 # 34 # 35	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.0632886 .00068795 .00068902 .0006997 .00069678	Temp. Temp. Temp. Temp.	DEG. C DEG. C	127933.131493 17.4741558483 17.5008715397 17.7674501235 17.6945796016
		BLOCK #7				
T. C. T. C. T. C.	# 38 # 39	Volts B.C. Volts B.C. Volts B.C. Volts B.C.	.0006945 .00068188 .00068375 .00069482	Temp. Temp.	DEG. C	17.63767324 17.3225732021 17.3895283267 17.6456604925

BLOCK #8

Volts D.C.

T. C.

Volts D.C. .00069182

Τ.	c.	#	43	Volts [D.C.	.00067 9 8	Temp.	DEG.	С	17.2706198805
۲.	С.	#	44	Volts (D.C.	.00057404	Temp.	DEG.	С	17.1267207233
Τ.	c.	#	45	Volts (D.C.	.00067405	Temp.	DEG.	С	17.1269705844
Τ.	C.	#	46	Volts [D.C.	.00068794	Temp.	DEG.	С	17.4739061621
Τ.	C.	#	47	Volts (D.C.	.00068	Temp.	DEG.	С	17.2756156288
Τ.	С.	#	48	Volts (D.C.	.00071555	Temp.	DEG.	С	18.1628100985

.00072565

Temp. DEG. C 17.570774941)

Temp. DEG. C 18.4145780413

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.00063946	TEMP. DEG. C	15.2519481857
VOLTS D.C.	.00063674	TEMP. DEG. C	16.1938628959
VOLTS D.C.	.00063796	TEMP. DEG. C	15.2244023031

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BLOCK # 1 HEATER VLOTS D.C. 1.479227
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BLOCK # 2 HEATER VLOTS D.C. 1.486967

BLOCK # 3 HEATER VLOTS D.C. 1.479592

BLOCK # 4 HEATER VLOTS D.C. 1.479553

BLOCK # 5 HEATER VLOTS D.C. 1.47892

BLOCK # 6 HEATER VLOTS D.C. 1.479432

BLOCK # 7 HEATER VLOTS D.C. 1.480182

BLOCK # 8 HEATER VLOTS D.C. 1.483219

INPUT VLOTAGE D.C. VLOTS 1.762348

BLOCK # 1 HEATER POWER WATTS .207325845281
BLOCK # 2 HEATER POWER WATTS .202714088825
BLOCK # 3 HEATER POWER WATTS .207110651253
BLOCK # 4 HEATER POWER WATTS .20713375774
BLOCK # 5 HEATER POWER WATTS .20750858305
BLOCK # 6 HEATER POWER WATTS .207205437481
BLOCK # 7 HEATER POWER WATTS .206760908026
BLOCK # 8 HEATER POWER WATTS .204955166461

T. C. # 21

T. C. # 22

T. C. # 23

T. C. # 24

Spacing: 11.913 mm

Power: 0.5 watts

Date: 17 November 1987

BLOCK #1

Volts D.C.

Volts D.C.

Volts D.C.

Volts D.C.

T. T. T.	c. c. c. c.	# # #	2 3 4	Volts Volts Volts	D.C. D.C. D.C.	.0007757 .00075667 .0007762 .00078484 .00078316	Temp. Temp. Temp. Temp.	DEG. DEG. DEG. DEG.	0000	19.6603067034 19.4357857992 19.6727356515 19.8874582327 19.8457139695 82326.501046
				BLO	OCK #2					
T. T. T.	0.0000000000000000000000000000000000000	# #	8 9 10 11	Volts Volts Volts Volts Volts	D.C. D.C. D.C.	.00077145 .00076157 .00076161 .00078706 .00077903 .0008491	Temp. Temp. Temp.	DEG. DEG. DEG. DEG.	0000	19.5546479536 19.3114222554 19.3099297019 19.9425148528 19.7430775733 21.4815142779
				BL	OCK #3					
T. T. T.	c. c. c. c.	# #	16	Volts Volts Volts Volts Volts	D.C. D.C. D.C.	.00076889 .00075901 .00075925 .00078362 .00077795	Temp. Temp. Temp.	DEG. DEG. DEG. DEG.	0000	19.4909931345 19.2452480378 19.2512190091 19.8571442991 19.7162344944 21.8790944579
				BL	OCK #4					
	C.			. Volts Volts		.00077507	•			19.5446457813 19.1414908466

Para secson recessor parameter recessor recessed recessor bessesse recessor recessor

.0007536

.0007804

.0007729

.00087154

Temp. DEG. C 19.1106331706

Temp. DEG. C 19.5905937854

Temp. DEG. C 22.0369474692

Temp. DEG. C 19.777126406

T. C. T. C. T. C. T. C. T. C.	# 27 # 28	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00075757 .00075131 .00075303 .00077823 .00075679	Temp. Temp. Temp. Temp.	DEG. C DEG. C DEG. C DEG. C DEG. C	19.2094206886 19.053540927 19.0964479449 19.723193952 19.4387701379 21.5065273003
		8L0CK #6				
T. C. T. C. T. C. T. C.	_	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.0522216 .00074305 .00074401 .00076745 .00075893	Temp. Temp. Temp.	DEG. C DEG. C DEG. C DEG. C	119433.511569 18.8480158435 18.871918566 19.4551836773 19.2432576979

Temp. DEG. C 21.3296751022

8L00k #7

Volts D.C. .00084297

T. C. # 36

τ. ο	Э.	# .	37	Volts	D.C.	.00074382	Temp.	DEG.	С	18.8571579109
T. (Ο.	# :	38	Volts	D.C.	.00072871	Temp.	DEG.	С	18.4908309143
Τ. (Э.	# :	39	Voits	D.C.	.0007342	Temp.	DEG.	С	18.6276080094
T. (Ο.	# .	40	Volts	D.C.	.00075992	Temp.	DEG.	С	19.2678875873
7. 0	Э.	- # .	4 1	Volts	D.C.	.00075266	Temp.	DEG.	С	19.0872397726
T. 0	<u>.</u>	# /	42	Volts	D.C.	.00083567	Temp.	DEG.	С	21.1487936715

BLOCK #8

Τ,	С.	#	43	Volts	D.C.	.00072821	ĭemp.	DEG.	С	18.4783720998
Τ.	С.	#	44	Volts	D.C.	.00071818	Temp.	DEG.	С	18.2283818345
Τ,	C.	#	45	Volts	O.C.	.00071525	Temp.	DEG.	С	18.1553298798
Τ.	C.	#	45	Volts	D.C.	.0007481	Temp.	DEG.	С	18.9737411325
Τ.	C.	#	47	Voits	D.C.	.00072787	Temp.	DEG.	С	18.4698999262
Τ.	С.	#	48	Volts	D.C.	.00081745	Temp.	DEG.	С	20.6972895837

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.00063986	TEMP. DEG. C	16.271959944
VOLTS D.C.	.00063916	TEMP. DEG. C	16.2344392365
UGLIS D.C.	.00063874	TEMP. DEG. C	16.243926516

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BLOCK # 1 HEATER VLOTS D.C. 2.31381
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BLOCK # 2 HEATER VLOTS D.C. 2.325955
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BLOCK # 3 HEATER VLOTS D.C. 2.314405
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BLOCK # 7 HEATER VLOTS D.C. 2.315302

BLOCK # 8 HEATER VLOTS D.C. 2.32009

INPUT VLGTAGE D.C. VLOTS 2.756495

STATE OF THE PROPERTY OF THE P

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BLOCK # 1 HEATER POWER WATTS .507074897951
BLOCK # 2 HEATER POWER WATTS .495751976067
BLOCK # 3 HEATER POWER WATTS .506523574661
BLOCK # 4 HEATER POWER WATTS .506538454423
BLOCK # 5 HEATER POWER WATTS .507505519849
BLOCK # 6 HEATER POWER WATTS .506753605495
BLOCK # 7 HEATER POWER WATTS .505691757717
BLOCK # 8 HEATER POWER WATTS .501238216109
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Spacing: 11.913

Power: 1.0 watt

Date: 17 November 1987

T. C. T. C. T. C. T. C. T. C.	# 4 # 5	Volts D.C.	.00086707 .00085117 .0008672 .0008699 .00087838	Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C	21.926356748 21.5327772142 21.9295733868 21.9963757079 22.2061250329 4860.44901559
T. C. T. C. T. C. T. C. T. C.	# 9 # 10 _ <u>#</u> 11	Volts D.C.	.00025135 .00084486 .00084905 .00089124 .00087403 .00102538	Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C	21.7848036559 21.3764951463 21.4802759762 22.5240398348 22.0985408295 25.8278095143
T. C. T. C. T. C. T. C. T. C.	# 13 # 14 # 15 # 16 # 17 # 18	BLOCK #3 Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00085993 .00084233 .00084307 .000865 .00087456	Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C	
T. C. T. C. T. C. T. C. T. C.	# 20 # 21 # 22	BLOCK #4 Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00086349 .00083851 .0008377 .00088555 .00087317	Temp. DEG. C	21.8377671353 21.2191719321 21.1991002913 22.4081213593 22.0772684996 26.9869333575

Τ.	С.	#	25	Volts D.	C00084506	•			21.381449386
Τ.	С.	#	26	Volts D.	C00083412	Temp.	DEG.	С	21.1103787297
Τ.	C.	#	27	Volts D.	C00083973	Temp.	DEG.	C	21.2494017377
Τ.	С.	#	28	Volts D.	C00088712	Temp.	DEG.	С	22.4222110038
Τ.	C.	#	29	Volts D.	C00086534	Temp.	DEG.	С	21.883548591
T.	C.	#	30	Volts D.	C00103501	Temp.	DEG.	С	26.0641255333
				.					

BLOCK #6

Т.	c.	# 31	Volts D.C.	.0483189	Temp. DEG. C	40950.8457007
Τ.	C.	# 32	Volts D.C.	.00082127	Temp. DEG. C	20.7917902939
Τ.	C.	# 33	Volts D.C.	.00082191	Temp. DEG. C	20.8076626403
Τ.	С.	# 34	Volts D.C.	.0008673	Temp. DEG. C	21.9320477899
Τ.	C.	# 35	Volts D.C.	.0008495	Temp. DEG. C	21.4914205785
T.	С.	# 36	Volts D.C.	.00102794	Temp. DEG. C	25.890640057

BLOCK #7

Τ.	С.	# 37	Volts D.C.	.00082072	Temp. DEG. C	20.7781495851
		# 38	Volts D.C.	.00079664	Temp. DEG. C	20.1805620148
Τ.	C.	* 39	Volts D.C.	.00080792	Temp. DEG. C	20.4605857652
Τ.	C.	# 40	Volts D.C.	.00095499	Temp. DEG. C	21.6273642643
Τ.	C.	# 41	Volts D.C.	.00084054	Temp. DEG. C	21.2719489826
Τ.	С.	# 42	Volts D.C.	.0010133	Temp. DEG. C	25.5312082938

BLOCK #8

Τ.	c.	#	43	Volts D.C.	00079706	Temp.	DEG.	С	20.1909912969
Τ.	C.	#	44	Volts D.C.	0007804	Temp.	DEG.	С	19.777126406
Τ.	C.	#	45	Volts D.C.	00077589	Temp.	DEG.	С	19.6650297407
Τ.	C.	#	45	Volts D.C.	00083833	Temp.	DEG.	C	21.2147116387
Τ.	C.	#	47	Volts D.C.	00080045	Temp.	DEG.	C	20.275162387
J.	С.	#	48	Volts D.C.	00098359	Temp.	DEG.	С	24.8009625523

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.00053962	TEMP. DEG.	С	16.2659529138
VOLTS D.C.	.00063902	TEMP. DEG.	С	16.250935021
VOLTS D.C.	.00063946	TEMP. DEG.	С	16.2619481867

BLOCK # 1 HEATER VLOTS D.C. 3.30153

BLOCK # 2 HEATER VLOTS D.C. 3.31896

BLOCK # 3 HEATER VLOTS D.C. 3.30248

BLOCK # 4 HEATER VLOTS D.C. 3.30231

BLOCK # 5 HEATER VLOTS D.C. 3.30105

BLOCK # 5 HEATER VLOTS D.C. 3.30192

BLOCK # 7 HEATER VLOTS D.C. 3.30351

BLOCK # 8 HEATER VLOTS D.C. 3.31045

INPUT VLOTAGE D.C. VLOTS 3.9327

BLOCK # 1 HEATER POWER WATTS 1.03159737134
BLOCK #-2 HEATER POWER WATTS 1.00840520317
BLOCK # 3 HEATER POWER WATTS 1.03034106218
BLOCK # 4 HEATER POWER WATTS 1.03055594104
BLOCK # 5 HEATER POWER WATTS 1.03223179627
BLOCK # 5 HEATER POWER WATTS 1.03108173149
BLOCK # 7 HEATER POWER WATTS 1.02897794895
BLOCK # 8 HEATER POWER WATTS 1.0197660953

Spacing: 11.913 mm

Power: 2.0 watts

Date: 17 November 1987

BLOCK #1

T. C. T. C. T. C. T. C. T. C.	# 3 # 4 # 5	2 3 4 5	Volts Volts Volts Volts	D.C. D.C. D.C.	.00098045 .00097514 .0009913 .00098299 .00100831	Temp. Temp. Temp. Temp.	DEG. DEG. DEG.	0000	24.7237197749 24.5930678391 24.9905733571 24.7862037369 25.4086353635 -184.374127276
		•)CK #2	.0010143	remp.	DES.	Ü	.0413/4/2/2/3

Τ.	С.	#	7	Volts	D.C.	.00098243	Temp.	DEG.	C	24.7724284372
Ŧ.	С.	#	8	Volts	D.C.	.0009667	Temp.	DEG.	С	24.3853301675
T.	С.	#	9	Volts	D.C.	.00097125	Temp.	DEG.	С	24.497332499
T.	C.	#	10	Volts	D.C.	.00102098	Temp.	DEG.	С	25.7197969998
T.	C.	#	1.1	Volts	D.C.	.00101565	Temp.	DEG.	С	25.5889222746
Τ.	С.	<u>#</u>	12	Volts	D.C.	.00132052	Temp.	DEG.	C	33.018342373

BLOCK #3

Τ.	С.	#	13	Volts	D.C.	.00099028	Temp.	DEG.	C	24.965492916
Τ.	С.	#	14	Volts	D.C.	. 0009 5138	Temp.	DEG.	С	24.2543408025
Т.	С.	#	15	Volts	D.C.	.0009671	Temp.	DEG.	С	24.3951775629
Τ.	С.	#	16	Volts	D.C.	.00098953	Temp.	DEG.	С	24.9470505872
T.	С.	#	17	Volts	D.C.	.00101017	Temp.	DEG.	С	25.4543274978
Τ.	С.	#	18	Volts	D.C.	.00138144	Temp.	DEG.	С	34.489273804

T. C.	# 19	Volts D.C.	.00100216	Temp. DEG. C	25.2575258334
T. C.	# 20	Volts D.C.	.00096484	Temp. DEG. C	24.3395371535
T. C.	# 21	Volts D.C.	.00096499	Temp. DEG. C	24.3432302987
T. C.	# 22	Volts D.C.	.00103215	Temp. DEG. C	25.9939544242
T. C.	# 23	Volts D.C.	.00102539	Temp. DEG. C	25.8280549698
T. C.	# 24	Volts D.C.	.00141873	Temp. DEG. C	35.3874418393

T. (C. :	# 25	Volts D.C.	.00098718	Temp.	DEG. C	24.8892500791
T. (C. 1	# 26	Volts D.C.	.0009658	Temp.	DEG. (24.3631727971
T. (C. :	# 27	Volts D.C.	.00097725	Temp.	DEG. 0	24.6449883594
T. (C. 4	# 28	Volts D.C.	.00105606	Temp.	DEG. 0	26.5802816979
T. (C. :	# 29	Volts D.C.	.00102888	Temp.	DEG. C	25.9137113471
T. (C. :	# 30	Volts D.C.	.0013606	Temp.	DEG. 0	33.9865910549
			BLOCK #6				
τ. (C. 1	# 31	Volts D.C.	0368589	Temp.	DEG. C	14771.9525454
T. (C. 1	# 32	Units D.C	00094839	Temo	DER C	73 93/3616310

T. C. # 32 T. C. # 33 Temp. DEG. C 23.9343516318 .00094839 Volts D.C. .00095853 Temp. DEG. C 24.1841533835 T. C. # 34 Temp. DEG. C 26.0741841938 Velts D.C. .00103542 Temp. DEG. C 25.2870143219 T. C. # 35 Volts D.C. .00100336 T. C. # 36 Volts D.C. .00135645 Temp. DEG. C 33.8864260423

BLOCK #7

Τ.	С.	# 3	37	Volts	D.C.	.00094947	Temp.	DEG.	С	23.9609638506
Τ.	C.	# 3	38	Volts	D.C.	.00090844	Temp.	DEG.	С	22.9489207544
Τ.	С.	# 3	39	Volts	D.C.	.00092711	Temp.	DEG.	С	23.4095948418
Τ.	C.	# 4	40	Volts	D.C.	.00101354	Temp.	DEG.	С	25.5598887592
Τ.	С.	# 4	41	Volts	D.C.	.00098967	Temp.	DEG.	С	24.9504932085
Τ.	С.	# 4	42	Volts	D.C.	.00133622	Temp.	DEG.	С	33.3978533444

BLOCK #8

Τ.	c.	#	43	Volts	D.C.	.00091492	Temp.	DEG.	С	23.1088960739
Τ.	Ç.	#	44	Volts	D.C.	.00088503	Temp.	DEG.	С	22.3705459828
Τ.	С.	#	45	Volts	D.C.	.00087504	Temp.	DEG.	С	22.1235222652
Τ.	C.	#	45	Volts	D.C.	.00099407	Temp.	DEG.	С	25.0586774191
Τ.	C.	#	47	Volts	D.C.	.00092318	Temp.	DEG.	С	23.3127390151
T.	C.	#	48	Volts	D.C.	.00128974	Temp.	DEG.	C	32.2734383616

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.00053972	TEMP. D	EG. C	16.2684558519
VOLTS D.C.	.00063946	TEMP. D	EG. C	15.2519481867
VOLTS D.C.	.0005398	TEMP. D	EG. C	16.2704581933

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BLOCK # 1 HEATER VLOTS D.C. 4.63301
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BLOCK # 2 HEATER VLOTS D.C. 4.65723

BLOCK # 3 HEATER VLOTS D.C. 4.63437

BLOCK # 4 HEATER VLOTS D.C. 4.63386

BLOCK # 5 HEATER VLOTS D.C. 4.6326

BLOCK # 6 HEATER VLOTS D.C. 4.63347

BLOCK # 7 HEATER VLOTS D.C. 4.83566

BLOCK # 8 HEATER ULOTS D.C. 4.64555

INPUT VLOTAGE D.C. VLOTS 5.5173

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BLOCK # 1 HEATER POWER WATTS 1.02818040143
BLOCK #12 HEATER POWER WATTS 1.96194247827
BLOCK # 3 HEATER POWER WATTS 1.01565555539
BLOCK # 4 HEATER POWER WATTS 1.0165011307
BLOCK # 5 HEATER POWER WATTS 1.01694119801
BLOCK # 5 HEATER POWER WATTS 1.016125905069
BLOCK # 7 HEATER POWER WATTS 1.016125905069
BLOCK # 8 HEATER POWER WATTS 1.00482079827
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Spacing: No Wall

Power: 0.2 watts

Date: 18 November 1987

T. C. T. C. T. C. T. C. T. C. T. C.	# 1 # 2 # 3 # 4 # 5 # 6	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00070266 .00069842 .00070024 .0007085 .0007047 .0599001	Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C	17.8413079193 17.735506202 17.7809250032 17.9869950151 17.9922034951 101722.622549
T. C. T. C. T. C. T. C. T. C.	_	BLOCK #2 Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. BLOCK #3	.00070104 .00069642 .00069706 .0007076 .0007043	Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C	17.800887:133 17.6855948216 17.701567551 17.9645460343 17.8822243832 18.5969572571
T. C. T. C. T. C. T. C.	# 15 # 16 # 17	Volts D.C.	.00070065 .00069458 .00069496 .00070667 .00070201	Temp. DEG. C	17.791'556852 17.6356700652 17.6491548749 17.9413476832 17.8250900914 18.7685807612
· .	# 19 # 20 # 21 # 23 # 23	Volts D.C.	.0007092 .00069264 .00069146 .00070405 .00070015	Temp. DEG. C	18.0044546288 17.5912447834 17.5617879137 17.875987336 17.7786792154 18.775553632

T. C. T. C. T. C. T. C. T. C.	* 26 * 27 * 28 * 29	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C.	.00059694 .00069151 .0005919 .00070297 .00059891 .00073208	Temp. DEG. 3 Temp. DEG. 3 Temp. DEG. 3 Temp. DEG. 3	10.6985707848 10.5620361217 10.572770036 10.8490422884 10.7477360124 16.5747351180
		BLOCK #6			
T. C. T. C. T. C. T. C. T. C.	# 32 # 33 # 34 # 35	Volts D.C.	.0684059 .0006877! .00068907 .0006984 .00069629 .00073087	Temp. DEG. S Temp. DEG. C Temp. DEG. C Temp. DEG. C	175187,827844 17,4661633458 17,5021199013 17,50216990931 17,6621036954 18,5446493733

BLOCK #7

T. C. # 37	Volts D.C.	.00069103	Temp. DEG. C	17.55 853 95
T. C. # 38	Volts D.C.	.00068235	Temp. DEG. 3	17.3343119003
T. C. # 39	Volts D.C.	.00068515	Temp. DEG. C	17,4052376826
T. C. <u>-# 4</u> 0	Volts D.C.	.00069503	Temp. DEG. C	17.6509023569
T. C. # 41	Volts D.C.	.00069528	⊺emp. DEG. €	17.8571419421
T. C. # 42	Volts D.C.	.0007268	Temp. DEG. 0	18.4432365486

BLOC+ #8

T.	С.	# 43	Volts D.C.	.00068306	Temp. DEG.	С	17.3520443201
		# 44	Volts D.C.	.00058003	Temp. DFG.	С	17,2763649867
		# 45	Volts D.C.	.00057934	Temp. DEG.	С	17.2591294683
		# 46	Volts D.C.	.00069069	Temp. DEG.	С	17.5425651131
		# 47	Volts D.C.	.00068316	Temp. DEG.	С	17.3545417703
Τ.	C.	# 48	Volts D.C.	.00071856	Temp. DEG.	С	18.237855358

SEESSESSE CONTRACTOR DESCRIPTION

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.00064216	TEMP. DEG.	С	16.3295236403
VOLTS D.C.	.00064005	TEMP. DEG.	С	16.2767154582
VOLTS D.C.	.00064024	TEMP. DEG.	С	16.2814709269

and the second of the second o

```
BLOCK # 1 HEATER VESTS 5.0. 1.471185
BLOCK # 3 HEATER LESTS 5.0. 1.478895
BLOCK # 3 HEATER LESTS 5.0. 1.478895
BLOCK # 4 HEATER LESTS 5.0. 1.471518
BLOCK # 4 HEATER LESTS 5.0. 1.471441
BLOCK # 5 HEATER LESTS 5.0. 1.471283
BLOCK # 7 HEATER LESTS 5.0. 1.471283
BLOCK # 7 HEATER LESTS 5.0. 1.471283
```

198 To 0.1988 000 TO 000 878 44

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Spacing: No Wall

Power: 0.5 watts

Date: 18 November 1987

7. 3			Volts D.C.	.00077196	Temp.	DEG. C	19.5673292028
T. C			Volts D.C.	.00075295		DEG. C	
T. C			Volts D.C.	.00075599		DEG. C	
T. C			Volts D.C.	.00078598	•	DEG. C	
T. S			Volts D.C.	.00077882		DEG. C	 - :
T. 0.		5	Valts D.C.	.050299	-	DEG. C	3, , 4 , 4 4 4 4 4 4 4 4
					C .D.	J20. J	30242.05 .25C
			EL001 #2				
• .	:	-	volts 5.1.	3035535	-		
	_	-	Valts D.C.	. 2027637			
• -	1	3		.0007576		QEG. C	19.2151432874
	•		volts 5.0.	.00075724		CEB. I	
	•	C.	Volts D.C.	. 0007866	ĭemp.	CEG. C	19.9311865124
	:		voits 0.1.	. 20077723		CEB. T	
	-	-	volts D.D.	.00085097	[™] ⊕™E.	CEGLO	01.5078045880
			€0.33k 4 3				
	:			0 00 76645	****	~ ≠ .	7 4237144648
	:	•	. • 5 _	22275425			424962374
•				322"-4"	***		10.00000
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T. C. T. C. T. C. T. C. T. C.	# 25 # 26 # 27 # 28 # 29 # 30	Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. Volts D.C. BLOCK #6	.00075665 .0007481 .00074975 .00077707 .00076375 .00084926	Temp. Temp. Temp. Temp.	DEG. C DEG. C DEG. C DEG. C DEG. C	19.1865296279 18.9737411325 19.0148127392 19.5943612718 19.3631613086 21.4854768222
T. C. T. C. T. C. T. C. T. C.	# 31 # 32 # 33 # 34 # 35 # 36	Volts D.C.	.0785367 .00073939 .00074344 .00076531 .00075892 .00084826	Temp. Temp. Temp. Temp.	DEG. C DEG. C DEG. C DEG. C DEG. C	306107.842144 18.7568760777 18.8577254643 19.401962028 19.2430099049 21.4607103931

BLOCK #7

T. C. # 37	Valts D.C.	.00074454	Temp. DEG. C	18.885:14364:
T. C. # 38	volts D.C.	.00072862		19.4885883509
T. C. # 39	volts 0.0.	.00073538	Temp. DEG. C	18.65700:4:38
T. C. <u>#</u> 40	⊍alts D.S.	.00275757	Temp, DEG. 0	19.2169849347
T. S. * 41	Valts 5.5.		Temp. DEG. 1	19, 350192789
T. C. # 42	.51 :: 5.5.	. 20082778	T emp. 0E 3. 0	59 . 9851. 55

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7. 0.		2011: 1.L	38377084	*#### 0 8 5 1	and the second section of the pro-
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	F. S. Carlotte	nut.	

BLOCK # 1 HEATER VLOTS D.C. 2.318752

BLOCK # 2 HEATER VLOTS D.C. 2.330904

BLOCK # 3 HEATER VLOTS D.C. 2.3193

BLOCK # 4 HEATER VLOTS D.C. 2.319161

BLOCK # 5 HEATER VLOTS D.C. 2.318535

BLOCK # 6 HEATER VLOTS D.C. 2.318915

BLOCK # 7 HEATER VLOTS D.E. 2.320085

BLOCK # 8 HEATER VLOTS D.C. 0.304893

INPUT NESTHER 1000 LOTS I TRITE

8200k : HEHTSHIR AND ANT 1 50 4 4222

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and the second second

Spacing: No Wall

Power: 1.0 watt

Date: 18 November 1987

8100- 1

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T. C. # 25 T. C. # 26 T. C. # 27 T. C. # 28 T. C. # 29 T. C. # 30	Volts D.C.	.00084304 .00082799 .0009304 .00088298 .00085725	Temp. DEG. C Temp. DEG. C Temp. DEG. C Temp. DEG. C	21.3314092579 20.9584242747 21.0181705619 22.3196664278 21.6833155213 25.9512579238
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BLOCK # 1 HEATER VLOTS D.C. 3.28433
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    ・ 日本の 1000 本の 400 本の 2000 できる場合を含めてまた。 1000 できません。 4000 本の 400 できる 400 を見るとしません。
    ・ 本 の 1000 本の 400 できません。 2000 を見るとしません。
    ・ ロ の 1000 では、400 を見ることをはません。 2000 を見るとしません。 4000 を見るとしません。
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Spacing: No Wall

Power: 2.0 watts

Date: 18 November 1987

T. C.		Volts D.C.	.00100754	Temp. DEG. (25.3921753126
T. C.	# 2	Volts D.C.	.00099476	Temp. DES. (25.0756404799
T. C.	* 3	Volts D.C.	.00100175	Temp. DEG. (25.2476959383
T. C.	x 4	Volts D.C.	.00104886	Temp. DEG.	26.4037952168
T. C.	# 5	Volts D.C.	.00102275	Temp. DEG. 0	25.7632503868
T. C.	= 5	valts D.C.	. 0058052 '	Temp. DEG. :	134.573569863
		8_00% #2			
T. D.		yolts D.C.	.00099972	Temp. SEG. 3	
T. C.		volts E.C.	. 00095:3:	Temp. DEG. (24.2526170225
T. S.		volts D.C.	.0009554	Temp. DEG. (24.1070536644
T. C.		Volts D.C.	.00102935	Temp. DEG. (
7. 0.		volts D.C.	.00102272	Temp. DEG. (
Ť. C.	# 12	Volts E.C.	.0013287	Temp. DEG.	33.2161121574
		BLGC+ #3			
T. C.	* 17	Volts D.C.	.00097605	Tama Med (24.6154607831
·		Volts D.C.	. 30094861	Temp. DEG. (
T. C.		Volts D.C.	.00094867	Temp. DEG. (
T. C.		Volts D.C.	.00102941	Temp. DEG. (
T. C.		Volts D.C.	.00098165	Temp. DEG. (
T. C.		Volts D.C.	.0013812	Temp. DEG. (
1	# 10	voits b.c.	.0013612	Temp. DEG. (. 54,4654677546
		BLOCK #4			
т. с.	# 19	Volts D.C.	.00097153	Temp. DEG. 0	24.5042241058
T. C.	# 20	Volts D.C.	.00093507	Temp. DEG. (23.6060143493
T. C.	# 21	Volts D.C.	.00093541	Temp. DEG. (23.6143980907
T. C.	# 22	Volts D.C.	.00098113	Temp. DEG. (24.740448554
т. с.	# 23	Volts D.C.	.00099116	Temp DEG. (
T. C.	# 24	Volts D.C.	.00138586	Temp. DEG. (

T. C. T. C. T. C. T. C. T. C.	# 26 # 27 # 28 # 29	Volts D.C.	.00097727 .00096117 .00095846 .00104858 .00101464	Temp. DEG. C	24.6454804705 24.2491694442 24.1824293543 26.3969315908 25.5641183855 33.926977279
		BLOCK #6			
T. C. T. C. T. C. T. C. T. C. T. C.	# 32 # 33 # 34 # 35	Volts D.C.	.0184661 .00093562 .00095092 .00102525 .00100942	Temp. DEG. C	1104.75486434 23.6195762117 23.9966909256 25.824618581 25.4359037688 34.1955430547

BLOCK #7

Τ.	С.	# 37	Volts D.C.	.00095197	Temp. DEG. C	24.0225605151
		# 38	Volts D.C.	.00090534	Temp. DEG. C	22.8723707565
		# 39	volts D.C.	.00093074	Temp. DEG. C	23.499232299
		- # 40	Volts D.C.	.00101508	Temp. DEG. C	25.5994819606
		# 41	Volts D.C.	.0009884	Temp. DEG. C	24.9192528199
Τ.	C.	# 42	Volts D.C.	.00133658	Temp. DE6. C	33.4065520005

BLOCK #8

T. C. T. C. T. C.	# 44	Volts D.C. Volts D.C. Volts D.C.	.00092359 .00090358	Temp. DEG. C	23.3228549035
T. C.	# 46	Volts D.C.	.00088635	Temp. DEG. C	22.4031775278 25.2354082889
T. C.		Volts D.C. Volts D.C.	.00093101 .0012997	•	23.5058914404 32.5146055682

BATH TEMPERATURES (TOP TO BOTTOM)

VOLTS D.C.	.0005428	TEMP. DEG. (C	16.3455401802
VOLTS D.C.	.00064058	TEMP. DEG. (С	16.2899805997
VOLTS D.C.	.00064043	TEMP. DEG. (С	16.2862263502

BLOCK # 1 HEATER VLOTS D.C. 4.64237

BLOCK # 2 HEATER VLOTS D.C. 4.66686

BLOCK # 3 HEATER VLOTS D.C. 4.64344

BLOCK # 4 HEATER VLOTS D.C. 4.64304

BLOCK # 5 HEATER VLOTS D.C. 4.64245

BLOCK # 6 HEATER VLOTS D.C. 4.64253

BLOCK # 7 HEATER VLOTS D.C. 4.54473

BLOCK # 8 HEATER VLOTS D.C. 4.65456

INPUT VLOTAGE D.C. VLOTS 5.52795

BLOCK # 1 HEATER POWER WATTS 2.03524258644
BLOCK # 2 HEATER POWER WATTS 1.98939924624
BLOCK # 3 HEATER POWER WATTS 2.03325203683
BLOCK # 4 HEATER POWER WATTS 2.0339963002
BLOCK # 5 HEATER POWER WATTS 2.0350927995
BLOCK # 6 HEATER POWER WATTS 2.03494500624
BLOCK # 7 HEATER POWER WATTS 2.03085070822
BLOCK # 8 HEATER POWER WATTS 2.01249809822

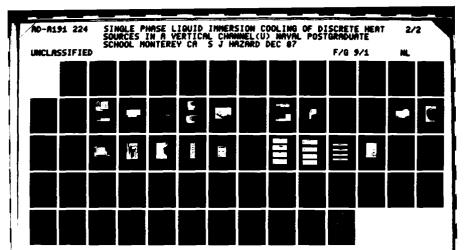
APPENDIX D

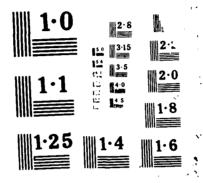
SOFTWARE

TEMPERATURE MEASUREMENT ACQUISION PROGRAM

```
10
                        REAL Volts(60)
  20
                        REAL Temp. 59 1
  40
                        PRINT "
                                                                                                                                                  BLOCK #1"
  50
                        PRINT
                        CUTPUT 709: "CONFMEAS DOV. 100, USE 0"
  51
 52
                        ENTER 709: Wolts: 60:
 50
                        GUTPUT 709: "CONFMEAS DOW, 100-105, USE 0"
  7.2
                        F09 1=0 T0 5
  30
                        ENTER 709: Volts IN
  3Ø
                         Temp I:=.0006797+:25525.1329*Volts I:--:507789.2467*(Volts I:-:0cts I
  21952034.3364+ \c.ts/I/M3//+ 8370810998.1974+\Volts/I/M4//
 100
                        PRINT "T. 3. #": [+! ]
                                                                                                             voits 0.0. ":volts(I), "Temp. DEG. C ":Temp(I)
  110
                        NEXT I
  120
                        PRINT " "
                        PRINT "
 130
                                                                                                                                                 BLOCH #2"
 140
 150
                        BUTPUT 709: "CONFMEAS DOU. 106-111 USE 0"
 160
                       FOR I=5 TO 11
 170
                        ENTER 709: Volts(I)
                        Temp(I) = .0005797 + (25925.1328 * Volts(I)) + (607789.2467 * (Volts(I) * Volts(I))) + (607789.2467 * (Volts(I) * Volts(I) * (Volts(I) * Volts(I) * (Volts(I) * Volts(I) * (Volts(I) * (Volts(I)
 180
  21952034.3364* Volts(I) 37 (+: 9370810996.1874*: Volts(I) 477
 190
                       PRINT "T. C. #":I+!,' volts D.C. ":Volts(I), "Temp. DEG. C ::Temp(I)
 200
                       NEXT I
 210
                       PRINT "
 220
                       PRINT "
                                                                                                                                                 BLOCE #3"
 230
                       PRINT " "
                       OUTPUT 709: "CONFMEAS DCV.112-117,USE 0"
 240
 250
                      FOR I=12 TO 17
 260
                       ENTER 709: Volts(I)
 270
                       Temp(I) = .0006797 + (25825.1328 * Volts(I)) + (607799.2467 * (Volts(I) * Volts(I)) + (607799.2467 * (Volts(I) * Volts(I))) + (607799.2467 * (Volts(I) * Volts(I) * (Volts(I) * Volts(I) * (Volts(I) * Volts(I) * (Volts(I) 
21952034.3364*(Volts(I)^3))+(8370910996.1874*(Volts\I)^4))
280
                       PRINT "T. C. \#":I+1," Volts D.C. ":Volts(I),"Temp. DEG. C ":Temp(I)
290
                      NEXT I
300
                      PRINT "
310
                      PRINT "
                                                                                                                                                BLOCK #4
320
                      OUTPUT 709: "CONFMEAS DCV.118-119,USE &
330
                      FOR I=18 TO 19
340
350
                      ENTER 709: Volts(I)
360
                      Temp(I)=.0006797+(25825.1328*Voits.I)=(607789.2467*/Voits/I)*Volts.I
21952034.3364+(Volts(I)/3))+(8370910958.1874+ Volts(I) 4))
370
                      PPINT "T. C. #":I+1," Volts C.C. ":volts-I-,""emp. BEG. C :Temp :
380
                      NEXT I
```

```
OUTPUT 709: "CONFMEAS DOU, 200+203, USE 0"
400
      FOR I=20 TO 23
410
      ENTER 709: Voits(I)
      Temp(I)=.0005797+:25325.1328+Valts I):- 607799.2467+ valts I:+Valts:I:+Valts:I:+Val
21952034.3364*(Voits(I))3:\+(83708)0996.1874*(Volts:I))4//
      PRINT "T. C. #":I+1," voits D.C. :voits(I),"Temp. DEG. C :Temp(I)
430
440
      NEXT I
      PRINT "
450
      PRINT "
                                      BLOCK #5"
450
      PRINT " "
470
480
      OUTPUT 709; "CONFMEAS DOV.204-209.USE 0"
490
      FOR I=24 TO 29
      ENTER T09: Volts(I)
500
      Temp(I)=.0006797+(25825,1328*Volts(I))+(607789.2467*(Volts(I))*Volts(I)))>+<
510
21952034.3364*(Volts(I) 3))+(8370810996.1874*(Volts(I)"4))
      PRINT "T. C. #": [+1] " volts O.C. ": Volts(I), "Temp. DEG. C ": Temp(I)
520
530
      NEXT I
      PRINT " "
540
      PRINT "
                                       BLCC+ #5"
550
      PRINT
560
      OUTPUT 709: "CONFMEAS DOU, 2:0-2:5, USE 0"
570
      FOR 1=30 TO 35
580
590
      ENTER 709: Volts (I)
      Temp(I)=.0008797+(25825.1328+valts(I))-(607789.2467+(valts(I)+valts(I))-(
600
21952034.3364*(Volts(I)^3))+(8370810996.1674*(Volts(I)^4))
      PRINT "T. C. #":I+1," Voits D.J. Tivoits(I), Temp. DEG. D. :Femp.I
610
620
      NEXT I
     FOR J=1 TO 14
530
     PRINT
640
      NEXT J
650
      PRINT "
660
                                      BL00K #7"
570
      OUTPUT 709; "CONFMEAS COU. 218-219 USE &"
680
690
      FOR 1=35 TO 39
700
      ENTER 709:Volts(I)
      Temp(I)=.0006797+/25825.1328+/blts I:>--607788.2467+/blts/I:>-vblts/I:>-/
710
21952034.3364*(Volts(I)^3))+:8370913998.1874**Volts(I) 4 **
720
      PRINT "T. C. #":I+1," volts D.C. ":volts(I , Temp. DEG. C :Temp:I)
730
      NEXT I
      OUTPUT 709: "CONFMEAS DD. 300-30" JUSE 0
740
      FOR I=48 TO 41
750
760
      ENTER 708: Volts/I
      Temp: I = .0006797+ 05805.1308+volts [ ---807789.0487+ loits [ + oits ]
770
21952034.3364*(volts.I) 2 (*.8373813988.1874* us.ts I 4
      PRINT T. C. # (I+')
                            voits 0.0. Fib.ts 1 Fempl 089 0
780
790
      NEXT I
```





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CALL CONTROL SECTION SECTION SECTIONS SECTIONS

```
800
                                         PRINT " "
                                        PRINT "
                                                                                                                                                                                                                                                                        BLOCK #8"
810
820
                                         PRINT " "
                                         OUTPUT 709; "CONFMEAS DCV, 302-307, USE 0"
830
840
                                         FOR I=42 TO 47
                                         ENTER 709; Volts(I)
850
                                         Temp(I) = .0006797 + (25825.1328 + Volts(I)) - (507789.2467 + (Volts(I) + Volts(I))) - (507789.2467 + (Volts(I) + Volts(I) + Volts(I))) - (507789.2467 + (Volts(I) + Volts(I) + Volts(I) + Volts(I) + (Volts(I) + (Volts(I) + (Volts(I) + Volts(I) + (Volts(I) + (Volts(I
860
21952034.3364*(Volts(I)^3))+(8370810996.1874*(Volts(I)^4))
                                         PRINT "T. C. #":I+1," Volts D.C. ":Volts(I), "Temp. DEG. C ":Temp(I)
870
880
                                         NEXT I
                                         PRINT "
890
1130 PRINT " "
1140 PRINT "
                                                                                                                                                                                                   BATH TEMPERATURES (TOP TO BOTTOM)"
1150 PRINT " "
1160 OUTPUT 709; "CONFMEAS DCV.317-319,USE 0"
1170 FOR I=57 TO 59
1180 ENTER 709; Volts(I)
1190 - \mathsf{Temp}(I) = .0006797 + (25825.1329 + \mathsf{volts}(I)) + (507789.2467 + (\mathsf{volts}(I) + \mathsf{volts}(I))) + (50778
21952034.3364*(Volts(I)^3))+(8370810996.1874*(Volts(I)^4)
1200 PRINT "UOLTS D.C. "; Uolts(I), "TEMP. DEG. C "; Temp(I)
1210 NEXT I
 1220 END
```

SINGLE THERMOCOUPLE TEMPERATURE VARIATION PROGRAM

```
10
                 REAL Volts(101)
20
                 REAL Temp(101)
30
                 PRINTER IS 1
40
                 PRINT " INPUT THE NUMBER OF CONSECUTIVE READINGS YOU WANT."
                 PRINT " "
50
                PRINT " THIS PROGRAM WILL READ ONLY THE THERMOCOUPLE SHOWN IN THE"
60
70
                 PRINT " SOURCE CODE. IF YOU WISH TO WORK WITH A DIFFERENT ONE YOU "
                 PRINT " MUST ENTER THAT T.C. ON THE CONFMEAS LINE OF THE CODE!!!"
80
90
                 INPUT N
                 PRINT"
100
                 PRINT " "
110
120
                 PRINT " INPUT THE NUMBER OF THE T.C. YOU WANT TO MEASURE."
130
                 INPUT M
131
                 PRINTER IS 701
140
                 PRINT " THESE ARE ": N, "CONSECUTIVE READINGS FOR THERMOCOUPLE # ":M
                 PRINT " "
150
                PRINT " "
160
170
                FOR I=50 TO N+50
180
                OUTPUT 709; "CONFMEAS DCV, 115, USE 0"
190
                 ENTER 709: Volts(I)
200
                Temp(I) = .0005797 + (25825.1328 + Volts(I)) - (607789.2467 + (Volts(I) + Volts(I))) + (607789.2467 + (Volts(I) + Volts(I) + Volts(I))) + (607789.2467 + (Volts(I) + Volts(I) + Volts(I) + (Volts(I) + Volts(I) + Volts(I) + (Volts(I) + (Volts(I) + Volts(I) + (Volts(I) + (Volts(I) + (Volts(I) + Volts(I) + (Volts(I) + (Volts(
21952034.3354*(Volts(I)^3))+(8370810995.1874*(Volts(I)^4))
210
                PRINT "VOLTS D.C. "; Volts(I), "TEMP. DEG. C "; Temp(I)
220
                NEXT I
230
                Evolts=0.
                Etemp=0.
240
250
                FOR J=50 TO N+50
260
                Evolts=Evolts+Volts(J)
270
                Etemp=Etemp+Temp(J)
280
                NEXT J
290
                Avolts=Evolts/(N+1)
300
                Atemp=Etemp/(N+1)
310
                PRINT " "
320
                 PRINT "AVERAGE VOLTAGE D.C. IS "; Avolts
330
                PRINT " "
340
                PRINT "AVERAGE TEMPERATURE DEG. C IS "; Atemp
350
                PRINTER IS 1
360
                 END
```

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HEATER VOLTAGE ACQUISION AND POWER CALCULATION PROGRAM

```
10
      REAL Volts(59)
920
      OUTPUT 709; "CONFMEAS DCV, 308-315.USE 0"
930
      FOR I=48 TO 55
940
      ENTER 709; Volts(1)
950
      PRINT " "
      PRINT "BLOCK #"; I-47, "HEATER VLOTS D.C."; Volts(I)
960
970
      NEXT I
980
      PRINT "
      PRINT " "
990
1000 PRINT " "
1010 OUTPUT 709; "CONFMEAS DCV, 316, USE 0"
1020 ENTER 709; Volts (56)
1030 PRINT " "
     PRINT " "
1040
     PRINT "INPUT VLOTAGE D.C. VLOTS"; Volts(56)
1050
1060 PRINT " "
1070 PRINT " "
1080 Resist=2.02
1090 FOR I=48 TO 55
1100 Pow=((Volts(56)-Volts(I))/Resist)+Volts(I)
1110 PRINT "BLOCK #"; I-47, "HEATER POWER WATTS" (Pow
1120 NEXT I
1220 END
```

```
DATA REDUCING PROGRAM WRITTEN IN FORTRAN
DOUBLE PRECISION T(48), TT(48), Q(8), QCONV(8), GR(8), TTT(8), AT(8)
     DOUBLE PRECISION TINF, KH20, BETA, NU, DXPG, DXR, KPG, KR, A, G, R1, R2, LEN DOUBLE PRECISION POWER, SPACE
     CHARACTER NAME×8, FNAME×8, EPNAME×8, GRNAME×8
     INTEGER RUN
     DXPG = 0.006731D0
     DXR = 0.003175D0
     KPG = 0.1421D0
     KR = 0.0389D0
     A = 0.000188D0
     G = 9.81D0
     LEN = 0.007874D0
     R1 = DXPG/(KPG*A)
     R2 = DXR/(KR*A)
     PRINT*, 'INPUT RUN #, INTEGER ONLY'
     READ*, RUN
     PRINT*, 'INPUT POWER VALUE IN WATTS'
     READ*, POWER
     PRINT*, 'INPUT SPACING IN MM' READ*, SPACE
     PRINTX, 'INPUT TEMPERATURE & INFINITY, DEG. C'
     READ*, TINF
     PRINT*, 'INPUT THE THERMAL CONDUCTIVITY OF H20, K'
     READ*, KH20
     PRINT*, 'INPUT THE EXPANSION COEFFICIENT, B'
     READ*, BETA
     PRINT*, 'INPUT THE VISCOSITY, NU'
     READ*, NU
*************
  READ IN TEMPERATURE DATA
**********
     PRINT*, 'INPUT THE NAME CORRESPONDING TO THE FILE TYPE OF DATA.' READ (5,'(A)') NAME REWIND 9
     OPEN (UNIT=9, FILE=NAME)
     DO 100 I=1,48,1
        READ(9, *) T(I)
 100 CONTINUE
***********
  CALCULATE T - TINF
**************
     DO 200 I=1,48,1
        TT(I) = T(I) - TINF
 200 CONTINUE
***********************

* CALCULATE CONDUCTION LOSSES THROUGH THE TEST SURFACE *

L = 6
     DO 300 I=1,8,1
        Q(I) = TT(L) \times (1.0/(R1+R2))
        L=L+6
 300 CONTINUE
```

SKKKKKI TSSEEDIJ DEDDENT DEDREIT KKKKKIK DEDDENT KRKKKKI KKKKKKT KKKKKKT KKKKKK TADERKKT TOPERKKT KK

ስ ነገር የመጀመር ያስፈር መስመር ያስተለር የሚያስፈር የሚያስ

```
************************
* CALCULATE CONVECTED HEAT FLUX *
*******************
     DO 400 I=1,8,1
        QCONV(I) = POWER - Q(I)
 400 CONTINUE
****************************
  CALCULATE AVERAGE SURFACE TEMPERATURES *
************************
     J = 1
     DO 105 I=1,8,1
        AT(I) = (T(J)+T(J+1)+T(J+2)+T(J+3)+T(J+4))/5.0
        J = J+6
 105 CONTINUE
*******************
  CALCULATE NONDIMENSIONAL TEMPERATURE *
***************
     DO 205 I=1,8,1
        TTT(I)=(AT(I)-TINF)/((QCONV(I)*LEN)/(A*KH2O))
 205 CONTINUE
*************
* CALCULATE MODIFIED GRASHOF NUMBER *
**************************************
     DO 500 I=1,8,1
        GR(I) = ((G \times BETA \times (QCONV(I) / A)) \times (LEN \times 4)) / (KH2O \times (NU \times 2))
 500 CONTINUE
*****************
* GENERATE OUTPUT DATA FILES *
     PRINT*, 'INPUT THE FILETYPE FOR THE OUTPUT FILE'
     READ (5, (A) ) FNAME
     REWIND 11
     OPEN (UNIT=11, FILE=FNAME)
     WRITE (11,1100) RUN, POWER, SPACE, TINF, KH20, BETA, NU
WRITE (11,1101)
1101 FORMAT(1X,/1X,/1X,/BLOCK #',3X,'QCOND',5X,'QCONV',5X,'GRASHOF #',4 CX,'NON-DIMEN. TEMP.')
     DO 600 J=1,8,1
        WRITE (11,1102) J,Q(J),QCONV(J),GR(J),TTT(J)
 600 CONTINUE
 1102 FORMAT(1X,3X,12,5X,F5.3,5X,F5.3,5X,E9.3,8X,F6.3)
1103 FORMAT(1X,/,1X,/,1X,/,5X,'B#',4X,'TF',6X,'TR',6X,'TL',6X,'TT',6X,'CTB',6X,'TH')
     K = 1
     DO 601 N=1,8,1
        WRITE(11,12)N,TT(K),TT(K+1),TT(K+2),TT(K+3),TT(K+4),TT(K+5)
        K = K+6
```

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Comment

```
601 CONTINUE
         WRITE(11,1104)
WRITE(11,1104)

1104 FORMAT(1X,/,1X,/,1X,'B# = BLOCK NUMBER',
    C3X,'TF = FRONT TEMP.',,
    C1X,'TR = RIGHT TEMP.',
    C4X,'TL = LEFT TEMP.',,
    C1X,'TT = TOP TEMP.',
    C6X,'TB = BOTTOM TEMP.',,
    C1X,'TH = HEATER TEMP.')
    PRINT*,'ENTER THE FILETYPE FOR EASYPLOT DATA FILE.'
    READ (5,'(A)') EPNAME
    REWIND 12
         REWIND 12
         OPEN (UNIT=12, FILE =EPNAME)
        K=1
         DO 700 M=1,8,1
          WRITE(12,12)M,TT(K),TT(K+1),TT(K+2),TT(K+3),TT(K+4),TT(K+5)
          K=K+6
  700 CONTINUE
    12 FORMAT (4X,12,2X,F6.3,2X,F6.3,2X,F6.3,2X,F6.3,2X,F6.3,2X,F6.3)
         PRINT*, 'ENTER THE FILETYPE FOR GR# EASYPLOT.'
READ(5,'(A)') GRNAME
REWIND 15
         OPEN(UNIT=15, FILE=GRNAME)
         DO 800 M=1,8,1
              WRITE(15,1300) TTT(M), GR(M)
  800 CONTINUE
1300 FORMAT(3X, F6.3, 2X, E10.3)
         STOP
```

END

APPENDIX E

TABULAR RESULTS

RUN NUMBER 1
POWER IN WATTS 0.200
SPACING IN MM. 73.810
AMBIENT TEMP. DEG. C 18.430
THERMAL CONDUCTIVITY OF H20 0.60038E+00
EXPANSION COEFFICIENT, B 0.18994E-03
VISCOSITY, NU 0.10453E-05

BLOCK #	QCOND	QCONV	GRASHOF #	NON-DIMEN.	TEMP.
1	0.003	0.197	0.114E+05	0.108	
2	0.003	0.197	0.114E+05	0.104	
3	0.003	0.197	0.114E+05	0.100	
4	0.004	0.196	0.114E+05	0.098	
5	0.003	0.197	0.114E+05	0.092	
6	0.003	0.197	0.114E+05	0.088	
7	0.003	0.197	0.114E+05	0.074	
8	0.003	0.197	0.115E+05	0.068	

B#	DTF	DTR	DTL	DTT	DTB	HTG
1	1.560	1.330	1.310	1.630	1.570	
2	1.450	1.320	1.340	1.530	1.485	2.200
3	1.350	1.200	1.220	1.560	1.530	2.400
4	1.350	1.260	1.190	1.500	1.400	2.420
5	1.240	1.150	1.170	1.460	1.330	2.160
6		1.110	1.150	1.380	1.330	2.080
7	0.960	0.840	0.890	1.220	1.150	2.030
8	1.000	0.800	0.790	1.120	0.940	1.730

B# = BLOCK NUMBER DTF = TF - TINF
DTR = TR - TINF DTL = TL - TINF
DTT = TT - TINF DTB = TB - TINF
DTH = TH - TINF

RUN NUMBER 2
POWER IN WATTS 0.500
SPACING IN MM. 73.810
AMBIENT TEMP. DEG. C 18.320
THERMAL CONDUCTIVITY OF H20 0.60035E+00
EXPANSION COEFFICIENT, B 0.18970E-03
VISCOSITY, NU 0.10460E-05

BLOCK # 1 2 3 4 5 6 7 8	QCOND 0.007 0.008 0.008 0.008 0.007 0.007	QCOI 0.49 0.49 0.49 0.49 0.49	93 92 92 92 93 93	GRASHOF \$ 0.285E+05 0.285E+05 0.285E+05 0.285E+05 0.285E+05 0.285E+05 0.286E+05	NON-	DIMEN. 0.099 0.095 0.093 0.091 0.089 0.095 0.072	TEMP.
B# 1 2 3 4 5 6 7 8	DTF 3.480 3.330 3.140 3.070 2.940 2.400 2.360	DTR 3.020 2.950 2.830 2.910 2.770 2.630 2.110 1.980	DTL 3.120 3.060 2.870 2.800 2.710 2.210 1.990	DTT 3.770 3.530 3.570 3.460 3.530 3.230 2.770	DTB 3.610 3.470 3.550 3.290 3.230 3.100 2.770 2.340	DTH 5.190 5.690 5.690 5.210 5.100 4.830 4.370	

RUN NUMBER 3
POMER IN WATTS 1.000
SPACING IN MM. 73.810
AMBIENT TEMP. DEG. C 16.080
THERMAL CONDUCTIVITY OF H20 0.59680E+00
EXPANSION COEFFICIENT, B 0.16480E-03
VISCOSITY, NU 0.11030E-05

BLOCK #	QCOND	QCONV	GRASHOF #	NON-DIMEN.	TEMP.
1	0.014	0.986	0.449E+05	0.080	
2	0.013	0.987	0.449E+05	0.076	
3	0.015	0.985	0.448E+05	0.078	
4	0.015	0.985	0.448E+05	0.077	
5	0.014	0.986	0.449E+05	0.076	
6	0.014	0.986	0.449E+05	0.072	
7	0.014	0.986	0.449E+05	0.064	
8	0.013	0.987	0.449E+05	0.059	

B#	DTF	DTR	DTL	DTT	DTB	DTH
1	5.490	5.190	5.260	6.220	5.690	
2	5.200	4.860	4.930	5.650	5.610	9.250
3	5.380	4.720	4.770	6.260	5.730	10.220
4	5.560	4.660	4.780	5.960	5.720	10.550
5	5.100	4.770	4.780	6.110	5.520	9.720
6		4.420	4.590	5.610	5.181	9.740
7	4.390	3.760	4.020	5.210	4.890	9.270
8	4.220	3.580	3.560	5.030	4.170	8.730

RUN NUMBER 4
POWER IN WATTS 2.000
SPACING IN MM. 73.810
AMBIENT TEMP. DEG. C 16.200
THERMAL CONDUCTIVITY OF H20 0.59696E+00
EXPANSION COEFFICIENT, B 0.16621E-03
VISCOSITY, NU 0.10998E-05

BLOCK # 1 2 3 4 5 6 7 8	QCOND 0.025 0.024 0.026 0.027 0.026 0.026 0.025	QCON 1.97 1.97 1.97 1.97 1.97 1.97	5 6 4 3 4 4 5	GRASHOF # 0.912E+05 0.912E+05 0.911E+05 0.911E+05 0.911E+05 0.912E+05 0.913E+05	NON	-DIMEN. 0.066 0.063 0.060 0.058 0.063 0.060 0.057 0.051	TEMP.
B# 1 2 3 4 5 6 7 8	DTF 8.980 8.540 8.160 8.110 8.270 7.750 7.070	7.920 7.750 7.330 7.940 7.390 6.660	DTL 8.770 7.980 7.510 7.520 7.960 7.860 7.190 6.130	DTT 10.030 9.580 9.700 8.530 10.120 9.430 9.360 8.900	DTB 9.340 9.350 8.520 8.840 9.460 8.930 8.590 7.270	DTH 16.660 18.030 18.210 17.550 17.790 17.270 16.070	

RUN NUMBER 5
POWER IN WATTS 0.200
SPACING IN MM. 11.913
AMBIENT TEMP. DEG. C 16.220
THERMAL CONDUCTIVITY OF H20 0.59699E+00
EXPANSION COEFFICIENT, B 0.16645E-03
VISCOSITY, NU 0.10992E-05

BLOCK # 1 2 3 4 5 6 7 8	QCOND 0.003 0.004 0.004 0.003 0.003 0.003		7 6 6 7 7	GRASHOF # 0.910E+04 0.909E+04 0.908E+04 0.908E+04 0.910E+04 0.910E+04 0.911E+04 0.913E+04		DIMEN. 0.129 0.122 0.120 0.117 0.109 0.100 0.093 0.075	TEMP.
B# 1 2 3 4 5 6 7 8	DTF 1.780 1.630 1.650 1.860 1.490 1.420	DTR 1.610 1.570 1.520 1.440 1.380 1.250 1.100	DTL 1.790 1.550 1.500 1.400 1.400 1.280 1.150	DTT 1.870 1.850 1.830 1.710 1.670 1.550 1.430	DTB 1.830 1.760 1.750 1.640 1.570 1.470 1.350	DTH 2.450 2.600 2.620 2.380 2.290 2.190	

RUN NUMBER 6
POWER IN WATTS 0.500
SPACING IN MM. 11.913
AMBIENT TEMP. DEG. C 16.250
THERMAL CONDUCTIVITY OF H20 0.59705E+00
EXPANSION COEFFICIENT, B 0.16685E-03
VISCOSITY, NU 0.10983E-05

BLOCK # 1 2 3 4 5 6 7 8	QCOND 0.008 0.008 0.008 0.008 0.008 0.007 0.007	QC0 0.4 0.4 0.4 0.4 0.4 0.4	92 92 92 92 92 93 93	GRASHOF # 0.229E+05 0.229E+05 0.229E+05 0.229E+05 0.229E+05 0.229E+05 0.229E+05 0.229E+05	NON-	DIMEN. 0.100 0.096 0.095 0.093 0.088 0.082 0.076	TEMP.
B# 1 2 3 4 5 6 7 8	DTF 3.410 3.300 3.240 3.390 2.960 2.620 2.230	DTR 3.190 3.060 2.990 2.890 2.800 2.600 2.240 1.980	DTL 3.420 3.060 3.000 2.860 2.850 2.620 2.380 1.900	DTT 3.640 3.690 3.610 3.530 3.470 3.210 3.020 2.720	DTB 3.600 3.490 3.470 3.340 3.190 2.990 2.840 2.220	DTH 5.230 5.630 5.790 5.260 5.080 4.900 4.450	

RUN NUMBER 7
POWER IN WATTS 1.000
SPACING IN MM. 11.913
AMBIENT TEMP. DEG. C 16.260
THERMAL CONDUCTIVITY OF H20 0.59706E+00
EXPANSION COEFFICIENT, B 0.16693E-03
VISCOSITY, NU 0.10981E-05

BLOCK # 1 2 3 4 5 6 7 8	QCOND 0.013 0.014 0.015 0.016 0.014 0.014 0.014	QC0 0.9 0.9 0.9 0.9	87 86 85 84 86 86	GRASHOF # 0.459E+05 0.458E+05 0.458E+05 0.458E+05 0.459E+05 0.459E+05 0.459E+05	иои	-DIMEN. 0.082 0.081 0.078 0.080 0.077 0.072 0.067 0.057	TEMP.
B# 12345678	DTF 5.670 5.520 5.490 5.580 5.120 4.520 3.930	DTR 5.270 5.120 5.050 4.960 4.850 4.530 3.920 3.520	DTL 5.670 5.220 5.070 4.940 4.990 4.550 4.200 3.400	DTT 5.740 6.260 5.610 6.150 6.160 5.670 5.370 4.950	DTB 5.950 5.840 5.850 5.820 5.620 5.010 4.010	DTH 9.570 10.340 10.730 9.800 9.630 9.270 8.540	

BLOCK # 2 3 4 5 6 7 8	QCOND 0.024 0.027 0.028 0.028 0.026 0.026 0.025	1.9 1.9 1.9 1.9 1.9	976 976 973 972 974 975	GRASHOF # 0.920E+05 0.920E+05 0.919E+05 0.918E+05 0.919E+05 0.919E+05 0.919E+05		N-DIMEN. 0.062 0.063 0.062 0.064 0.065 0.062 0.057	TEMP.
B# 12345678	DTF 8.450 8.500 8.690 8.990 8.620 7.690 6.840	DTR 8.320 8.110 7.980 8.070 8.090 7.660 6.680 6.100	DTL 8.720 8.230 8.120 8.370 7.910 7.140 5.850	DTT 8.520 9.450 8.680 9.720 10.310 9.800 9.390 8.790	DTB 9.140 9.320 9.180 9.560 9.640 9.640 8.680 7.040	DTH 16.750 18.220 19.120 17.720 17.620 17.130 16.000	

	B1 23 45 67 8	BLOCK 1 2 3 4 5 6 7 8	RUN NU POWER SPACIN AMBIEN THERMA EXPAUS VISCOS	B\$ 1234 567 8	BLOCK 1 2 3 4 5 6 7 8	RUN NI POWER SPACII AMBIEI THERM EXPAUS VISCOS
	1 1 1 1 1 1 1 1 1	•	IN IG I IT T L C		•	IN NG NT AL SIO
	DTI .50 .40 .70	000000	WATEMITEMITEMITEMITEMITEMITEMITEMITEMITEMI	DT8.4 8.5 8.6 8.6 7.6	00000	WAIN TEM CON
	60 00 00 00 00 00	COND .003 .003 .004 .003 .003 .003	MM. P. D DUCT DEFF	50 00 90 90 20 	CONI . 024 . 027 . 028 . 026 . 026 . 025	MM. IP. I
	1 1 1 1 1 1 1		N EG IV IC	8 8 7 6		DE(
	DTR .44 .39 .34 .29 .26 .17 .03		G. C	DTF 8.32 8.12 7.98 8.07 8.09 7.66 6.68		VIT'
	0000000	0.00.00.00.	HR O T,		1.1.1.1.1.	913 C Y (
	1 1 1 1 1	ONV 197 197 196 197 197	16. F H B	-	97 97 97 97 97 97	16)F B
		7	. 3 12 0	8. 8. 8. 7.	6632445	H2
90	TL 480 400 350 260 270 200 100 960		NG 1 00 0 (720 230 120 070 370 910 140 850		270 20 0.16
		0.0.0.0.0.0.) . 5		0.00.00.00.00.00.00.00.00.00.00.00.00.0	
	1.1.1.1.1.	91 91 91 91 91	97	8989099	999999	
	TT 661 661 581 551 431 351 241	HOI 8E 9E 9E 9E 9E		DTT.52.45.45.31.39.79	20E 20E 19E 19E 19E	
))))	+04 +04 +04 +04 +04	E+0	0 0 0 0 0 0	+ 05 + 05 + 05 + 05 + 05 + 05 + 05	E+0
	11111111		0	9		00
	DTB .590 .580 .520 .480 .360	ИС		DTB 9.14 9.32 9.18 9.56 9.64 9.02 3.68	N	
		I – N		0000000	0N-	
	2. 2. 2. 2. 2. 2.	0.0.0.0.0.0.		16 18 19 17 17	000000	
	TH 300 470 480 270 240 140 940	EN. 112 109 107 106 100 094 088 076		0TH .750 .220 .120 .720 .620 .130	MEN. .062 .063 .064 .065 .065	
		TE				
		EM			E	

PRODUCED TOTAL TOTAL STREET STREETS TOTAL STREET

RUN NUMBER 10
POWER IN WATTS 0.500
SPACING IN MM. NO SHROUDING WALL
AMBIENT TEMP. DEG. C 16.300
THERMAL CONDUCTIVITY OF H20 0.59712E+00
EXPANSION COEFFICIENT,B 0.16737E-03
VISCOSITY,NU 0.10970E-05

BLOCK # 1 2 3 4 5 6 . 7 8	QCOND 0.008 0.008 0.008 0.008 0.008 0.008 0.007		92 92 92 92 92	GRASHOF # 0.230E+05 0.230E+05 0.230E+05 0.230E+05 0.230E+05 0.230E+05 0.231E+05		DIMEN. 0.096 0.093 0.091 0.088 0.085 0.080 0.074 0.066	TEMP.
B#12345678	DTF 3.270 3.190 3.130 3.250 2.890 2.550 2.270	DTR 3.040 2.910 2.840 2.710 2.670 2.460 2.110	DTL 3.120 2.900 2.830 2.700 2.710 2.560 2.560	DTT 3.640 3.630 3.550 3.410 3.390 3.100 2.750	DTB 3.440 3.390 3.270 3.180 3.060 2.940 2.310	DTH 5.230 5.600 5.640 5.180 5.160 4.900 4.540	

RUN NUMBER 11
POWER IN WATTS 1.000
SPACING IN MM. NO SHROUDING WALL
AMBIENT TEMP. DEG. C 16.300
THERMAL CONDUCTIVITY OF H20 0.59712E+00
EXPANSION COEFFICIENT, B 0.16737E-03
VISCOSITY, NU 0.10970E-05

BLOCK # 1 2 3 4 5 6 7 8	QCOND 0.014 0.015 0.015 0.015 0.014 0.014	QC0 0.9 0.9 0.9 0.9	86 85 85 86 86 87	GRASHOF # 0.461E+05 0.461E+05 0.460E+05 0.461E+05 0.461E+05 0.461E+05		-DIMEN. 0.080 0.075 0.077 0.076 0.075 0.075 0.065 0.059	TEMP.
B#12345678	DTF 5.410 5.150 5.300 5.490 5.030 4.470 4.030	DTR 5.160 4.750 4.850 4.660 4.280 3.310 3.750	DTL 5.270 4.760 4.740 4.700 4.720 4.480 4.150 3.500	DTT 6.190 5.770 6.050 5.890 6.020 5.500 5.240 4.980	DTB 5.650 5.650 5.630 5.570 5.380 5.210 4.940 4.110	DTH 9.340 10.250 10.480 9.650 9.690 9.180 8.650	

RUN NUMBER 12
POWER IN WATTS 2.000
SPACING IN MM. NO SHROUDING WALL
AMBIENT TEMP. DEG. C 16.310
THERMAL CONDUCTIVITY OF H20 0.59714E+00
EXPANSION COEFFICIENT, B 0.16753E-03
VISCOSITY, NU 0.10966E-05

BLOCK # 1 2 3 4 5 6 7 8	QCOND 0.025 0.025 0.026 0.027 0.026 0.026 0.025	1.9 1.9 1.9 1.9	975 1775 1774 1774 1774 1775	GRASHOF # 0.924E+05 0.923E+05 0.923E+05 0.923E+05 0.924E+05 0.924E+05 0.925E+05		DIMEN. 0.063 0.063 0.063 0.063 0.063 0.065 0.057	TEMP.
B#12345678	DTF 9.080 8.570 8.300 8.190 8.330 7.710 7.010	DTR 8.770 7.940 7.630 7.300 7.310 6.560 6.520	DTL 8.940 7.800 7.460 7.300 7.870 7.690 7.190 6.090	DTT 10.090 9.590 9.620 8.430 10.090 9.510 9.290 8.920	DTB 9.450 9.480 8.440 8.680 9.250 9.130 8.610 7.200	DTH 16.910 18.170 18.290 17.620 17.890 17.100 16.200	

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APPENDIX F FIGURES

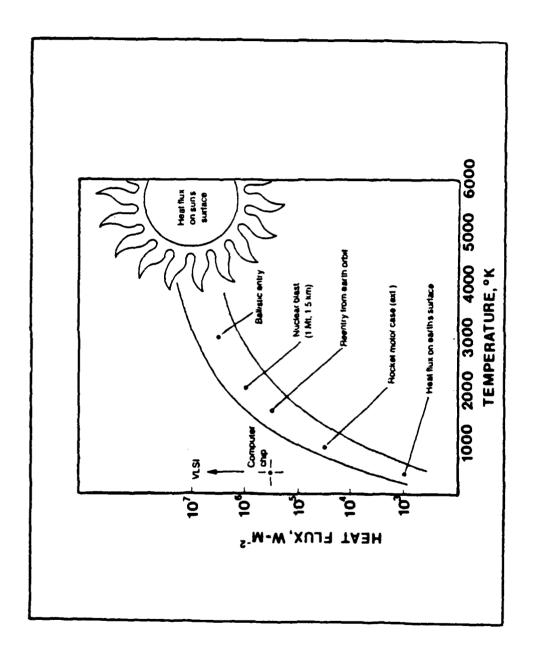


Figure 1. Temperature versus Heat Flux for Various Phenomena [Ref. 5]

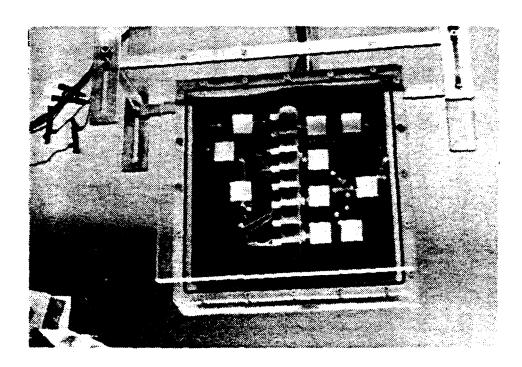


Figure 2. Assembled Test Surface and Shrouding Wall

PARAMENTAL MANAGEMENT RESERVED BESSESSA WORKSOM TANAMENTAL



Figure 3. Mounted Foil Heater

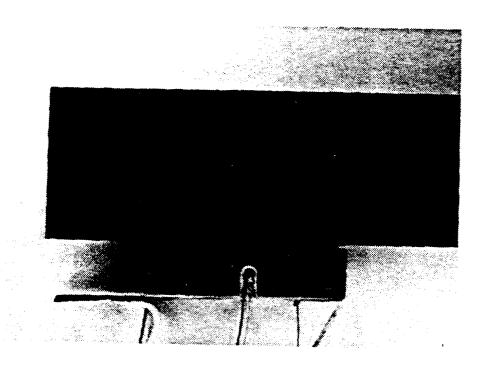


Figure 4. Mounted Thermocouple

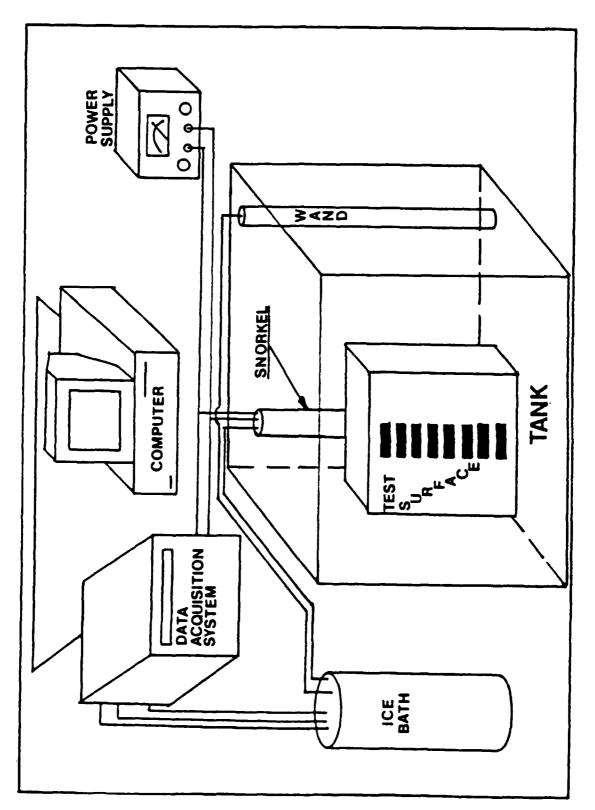


Figure 5. System Configuration

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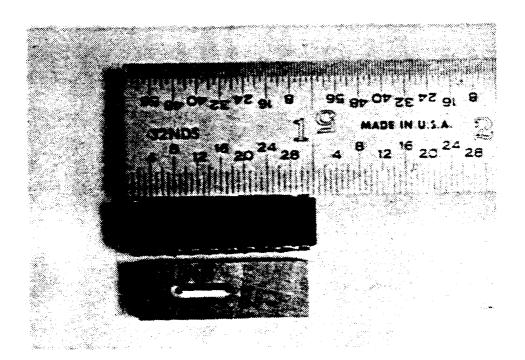


Figure 6. 20 Pin DIP and Chip Comparison, Too View

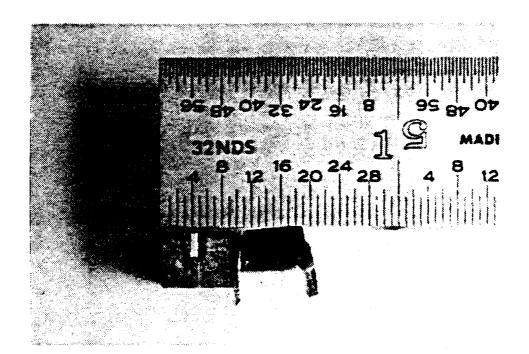


Figure 7. 20 Pin DIP and Chip Comparison, End View

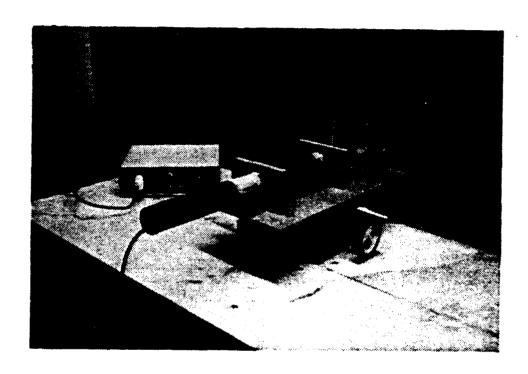


Figure 8. Laser and Cylindrical Lens

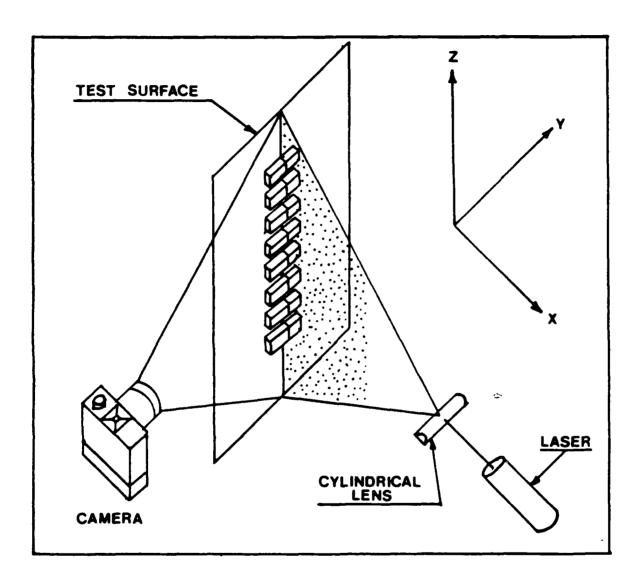


Figure 9. Laser and Camera Orientation

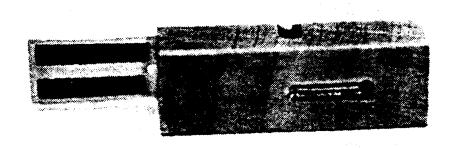


Figure 10. Block with Grooves

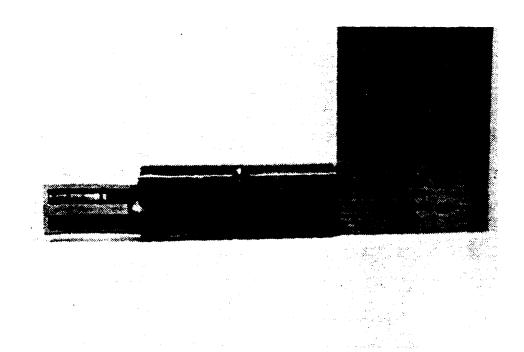


Figure 11. Mounted Foil Heater, End Measured

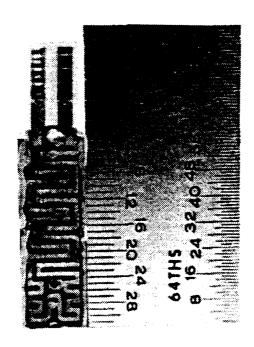
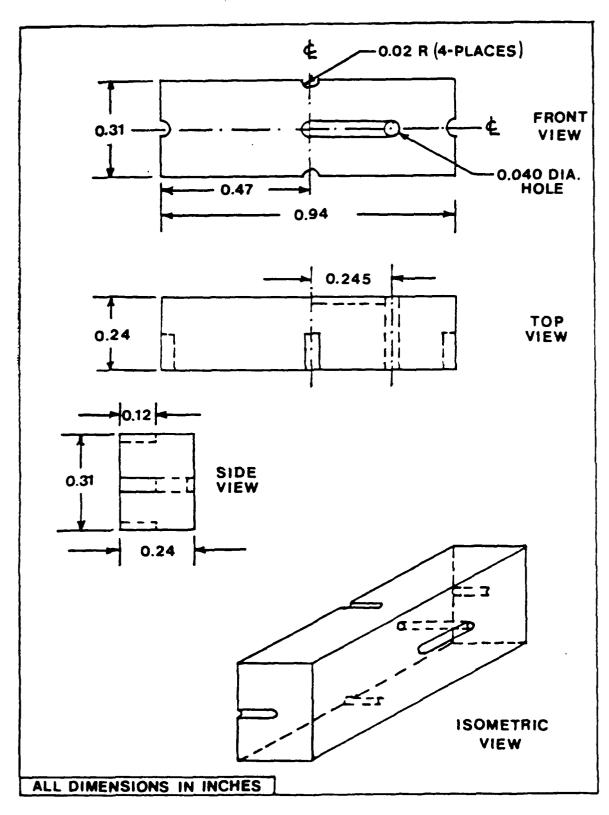


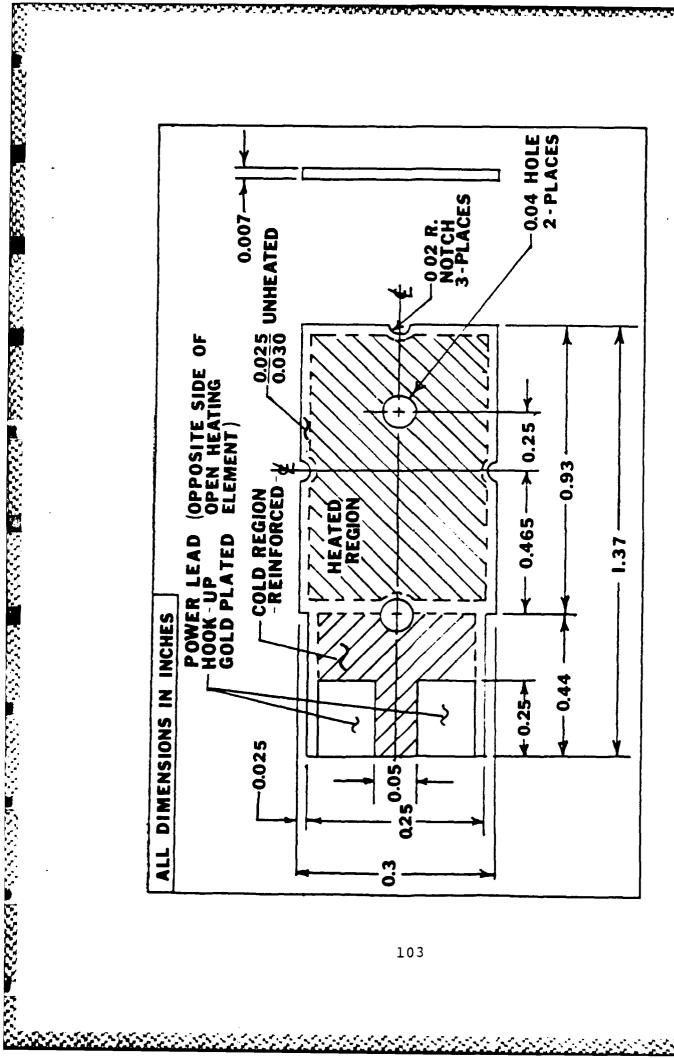
Figure 12. Mounted Foil Heater, Length Measured



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Figure 13. Heater Block Schematic



Foil Heater Schematic Figure 14

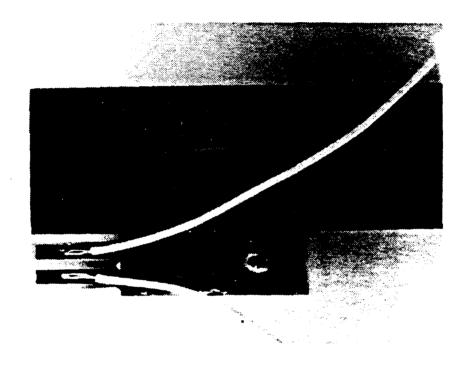


Figure 15. Power Lead Attachment

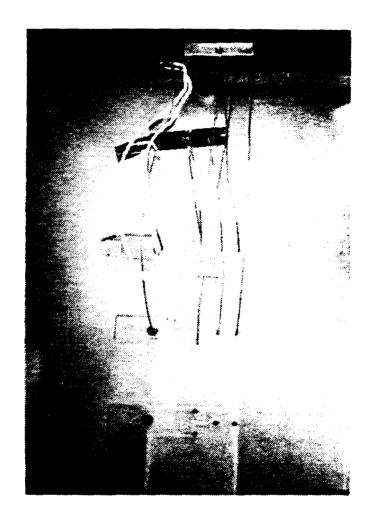


Figure 16. Slot and Holes in Test Surface

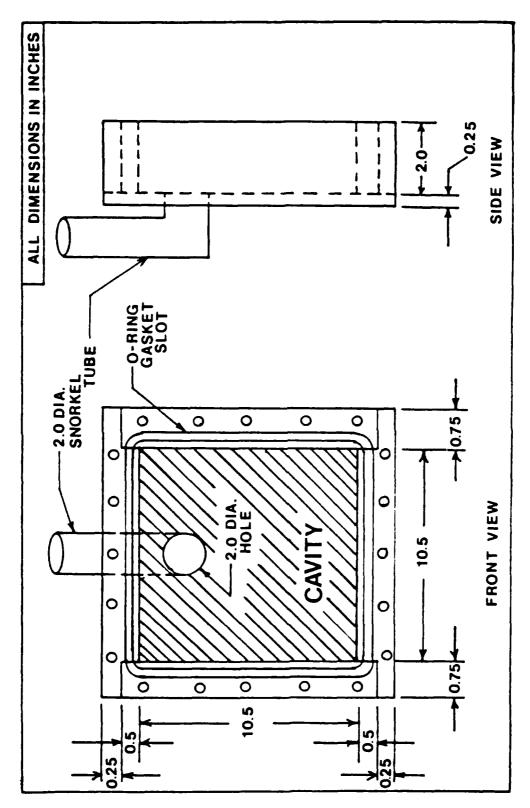
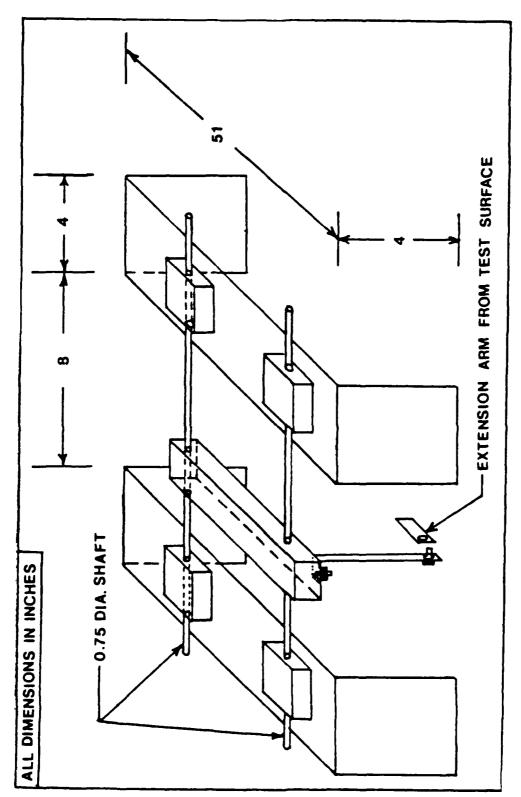


Figure 17. Containment Back Schematic

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Test Surface and Shrouding Wall Support Bracket Figure 18.

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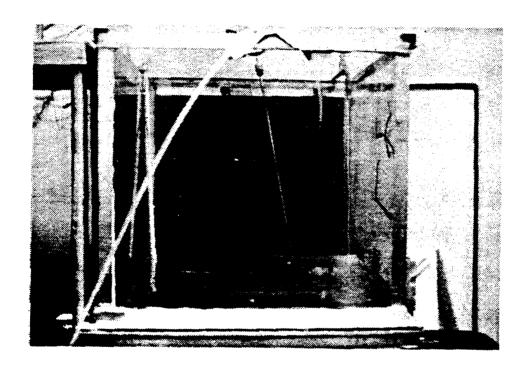


Figure 19. Immersion Tank

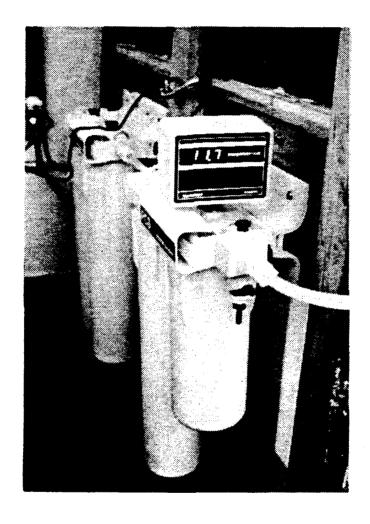
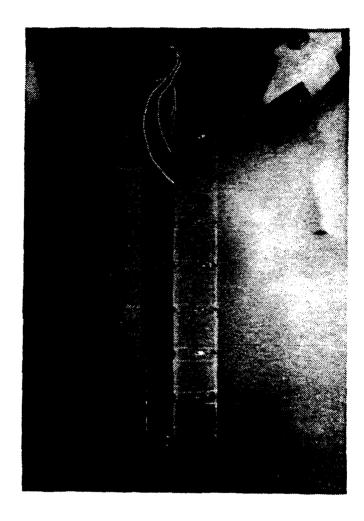


Figure 20. Filtration and Purification System



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Figure 21. Mounting the Heater Assemblies

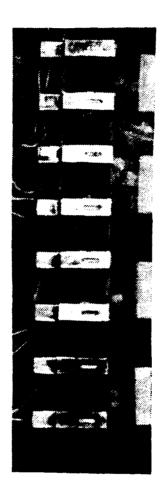


Figure 22. Mounted Heater Assemblies

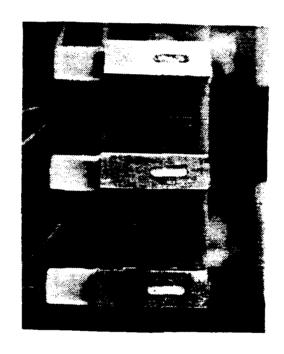


Figure 23. Close-Up of Mounted Heater Assemblies

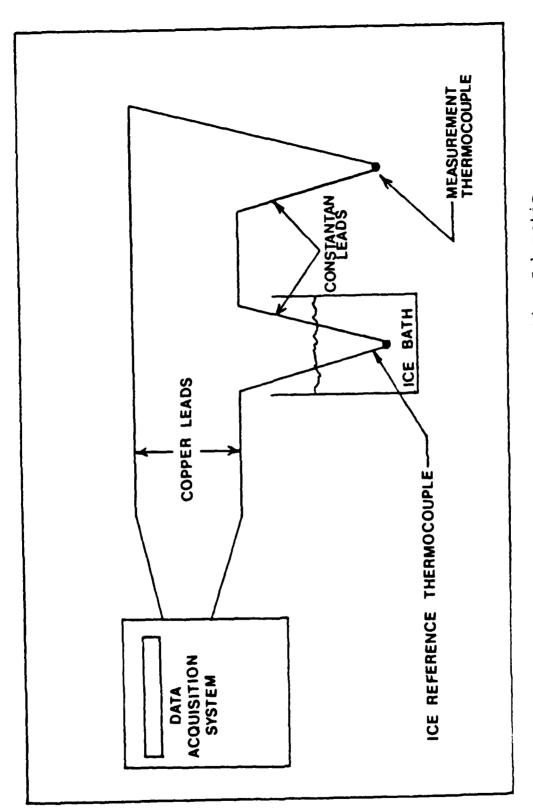
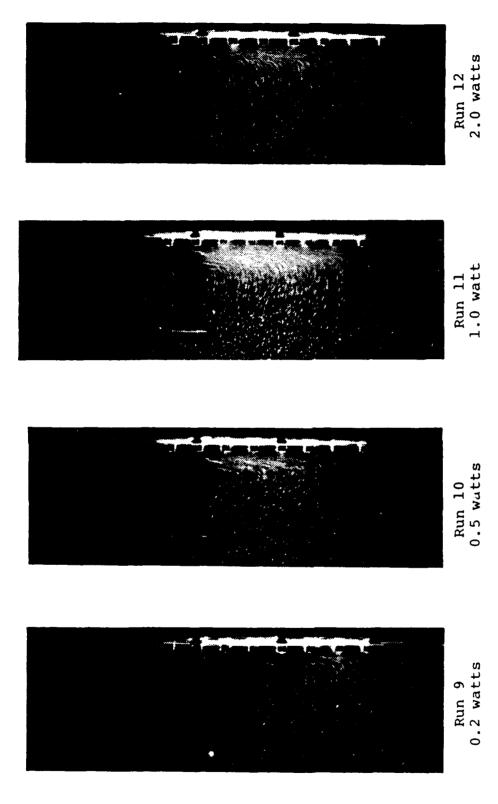
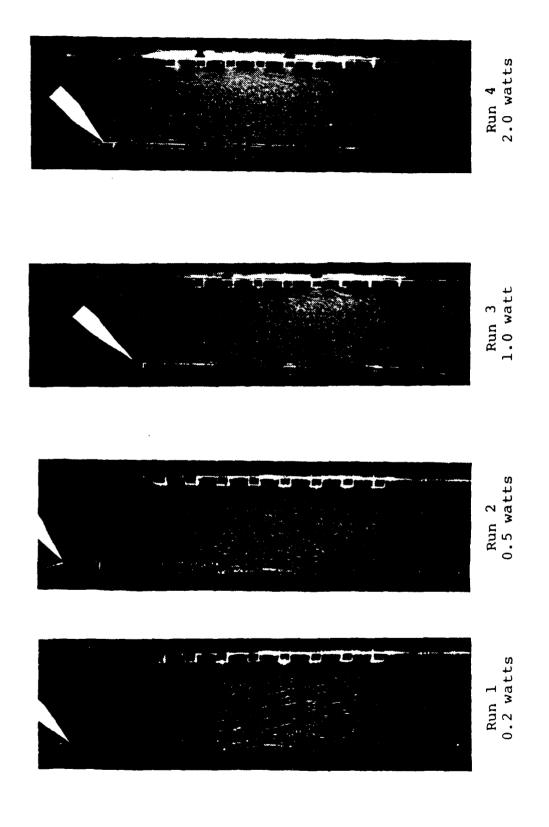


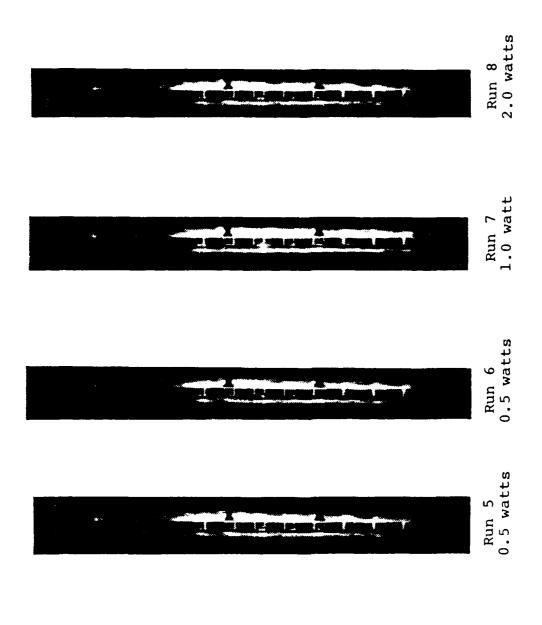
Figure 24. Thermocouple Connection Schematic



Flow Visualization Photographs for the No Wall Spacing Figure 25.



Flow Visualization Photographs for the 73.81 mm Spacing Figure 26.



Flow Visualization Photographs for the 11.913 mm Spacing Figure 27.

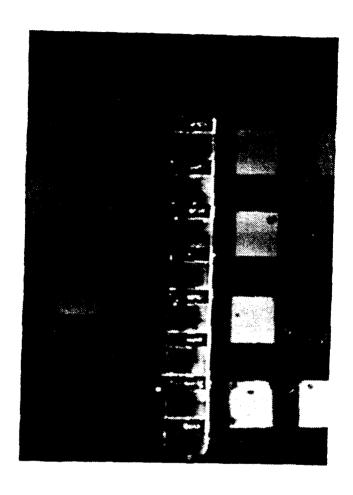
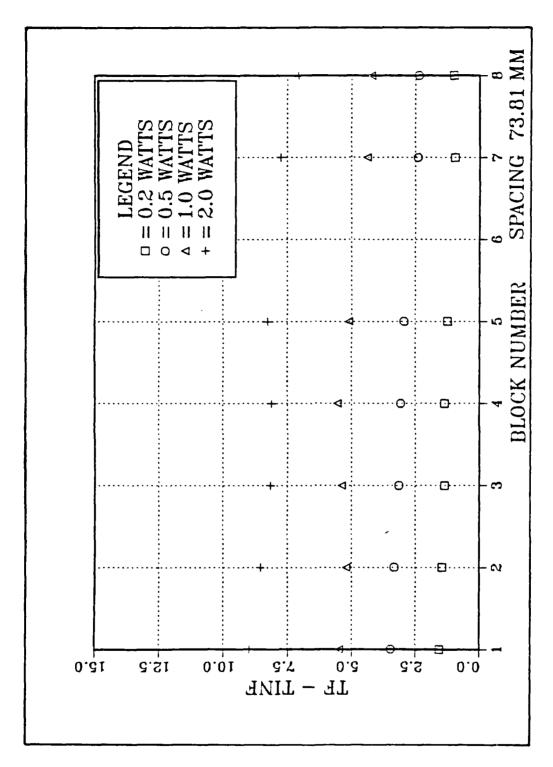
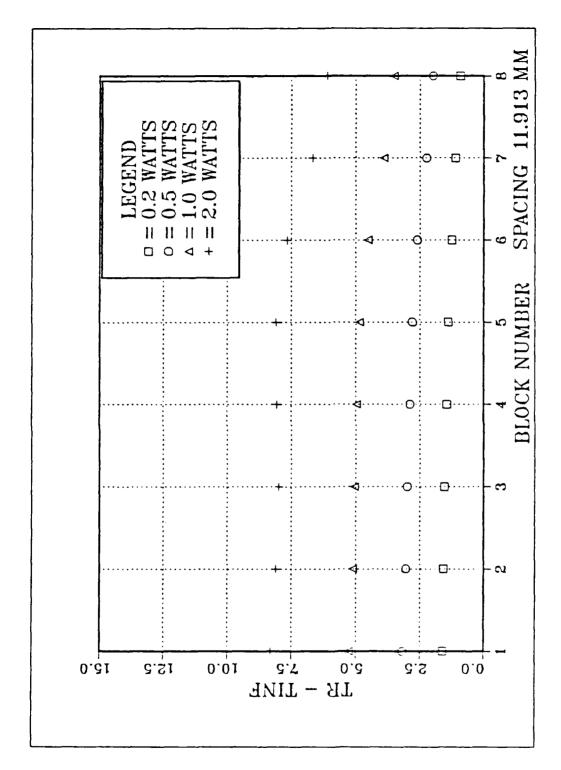


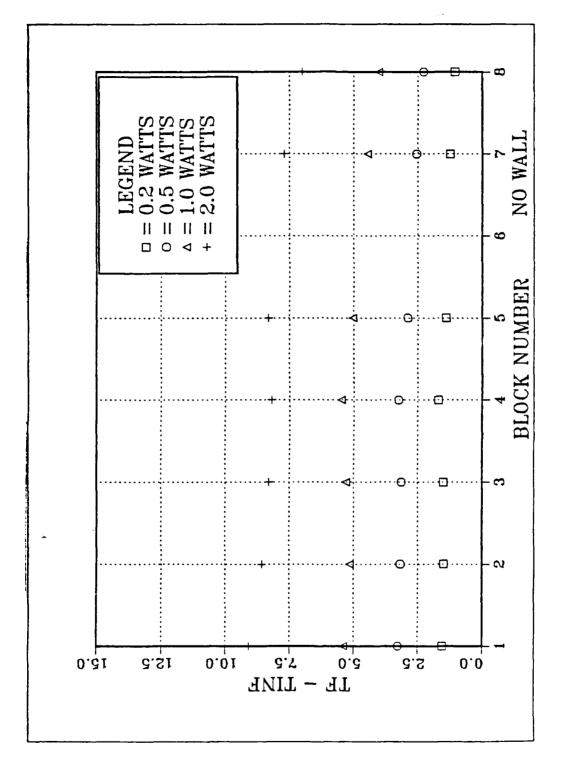
Figure 28. Across Test Surface Flow Visualization Photograph



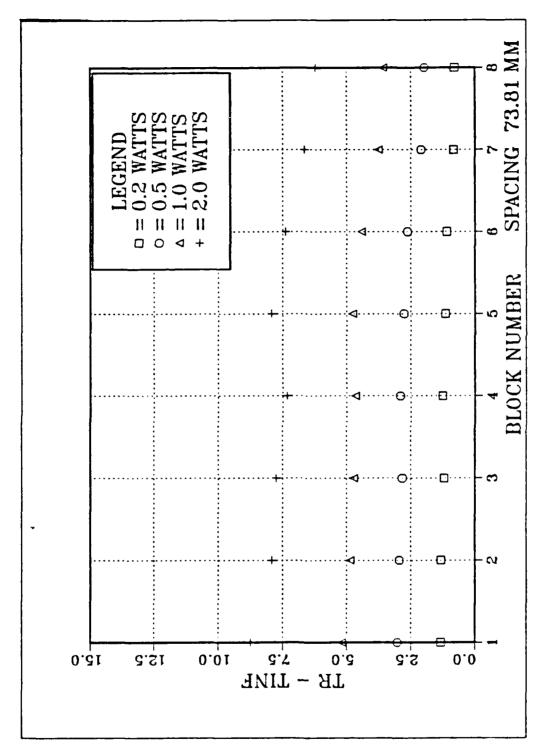
Block Number vs. Excess Temperature (Front Face) Runs 1-4 Figure 29.



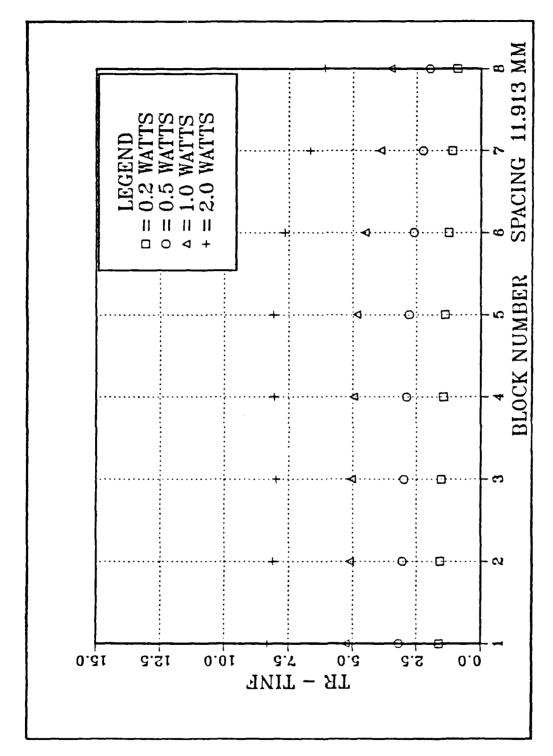
Block Number vs. Excess Temperature (Front Face) Runs 5-8 Figure 30.



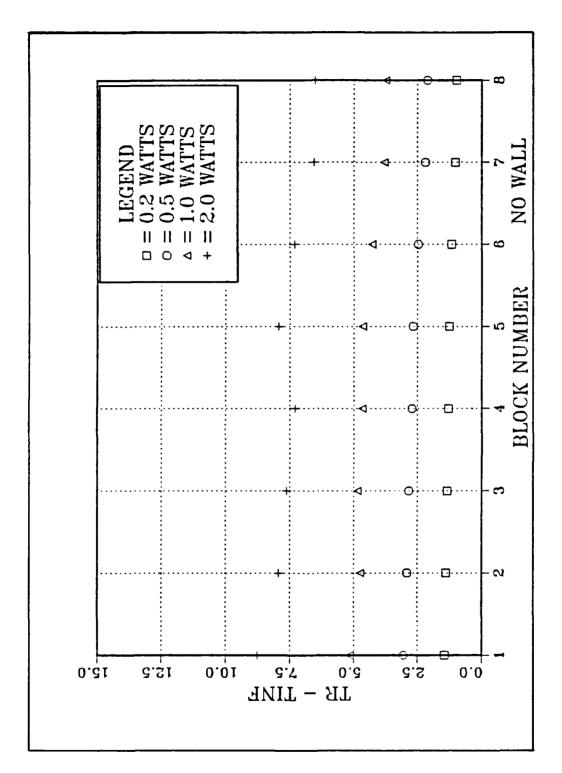
Block Number vs. Excess Temperature (Front Face) Runs 9-12 Figure 31.



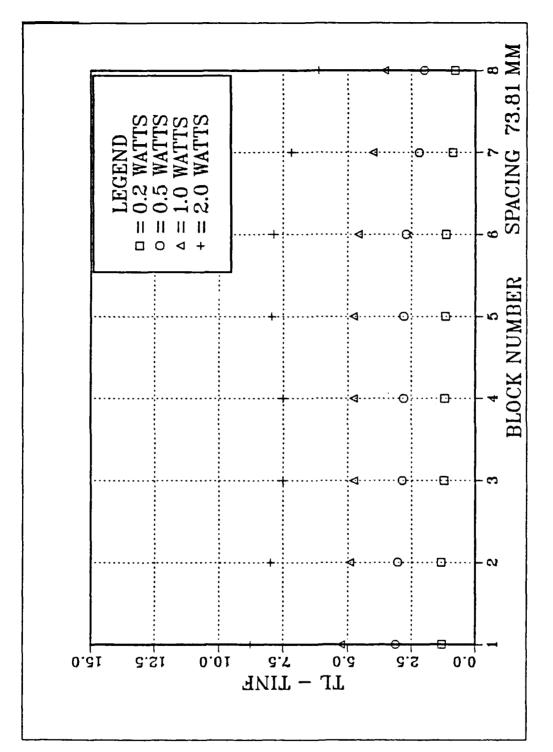
Block Number vs. Excess Temperature (Right Face) Runs 1-4 Figure 32.



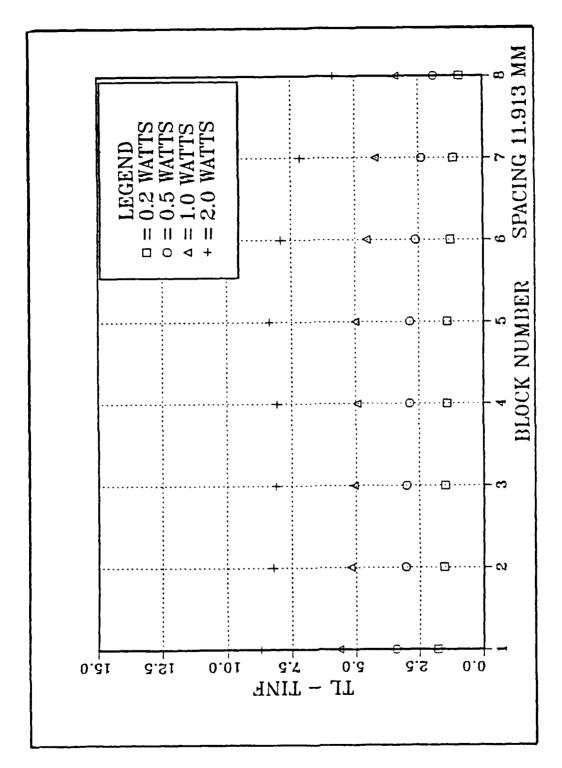
Excess Temperature (Right Face) Runs 5-8 Block Number vs. Figure 33.



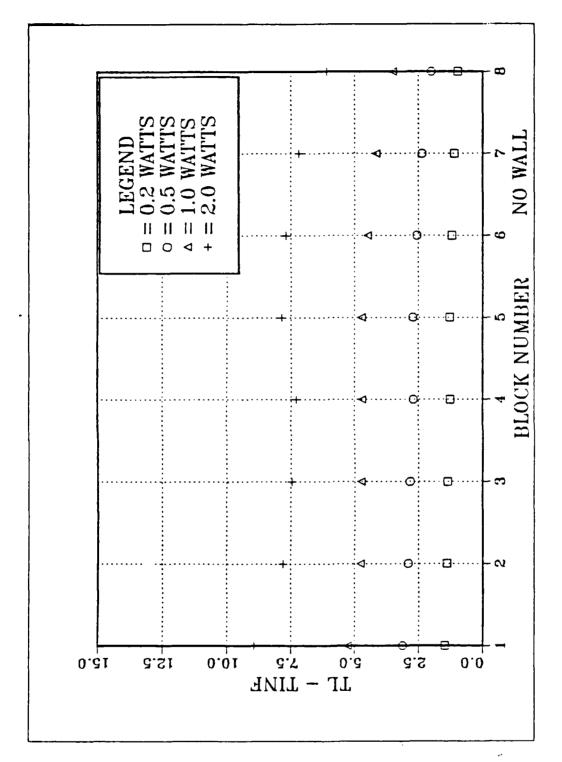
Block Number vs. Excess Temperature (Right Face) Runs 9-12 Figure 34.



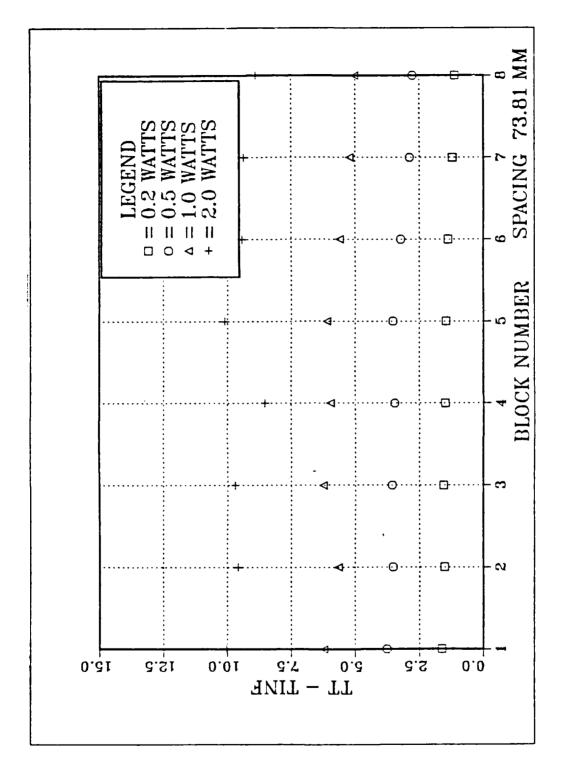
Block Number vs. Excess Temperature (Left Face) Runs 1-4 Figure 35.



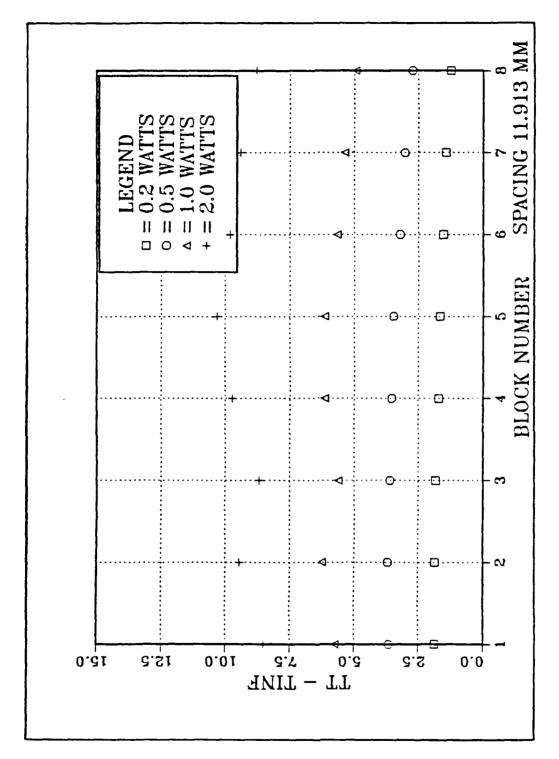
Block Number vs. Excess Temperature (Left Face) Runs 5-8 Figure 36.



Block Number vs. Excess Temperature (Left Face) Runs 9-12 Figure 37.

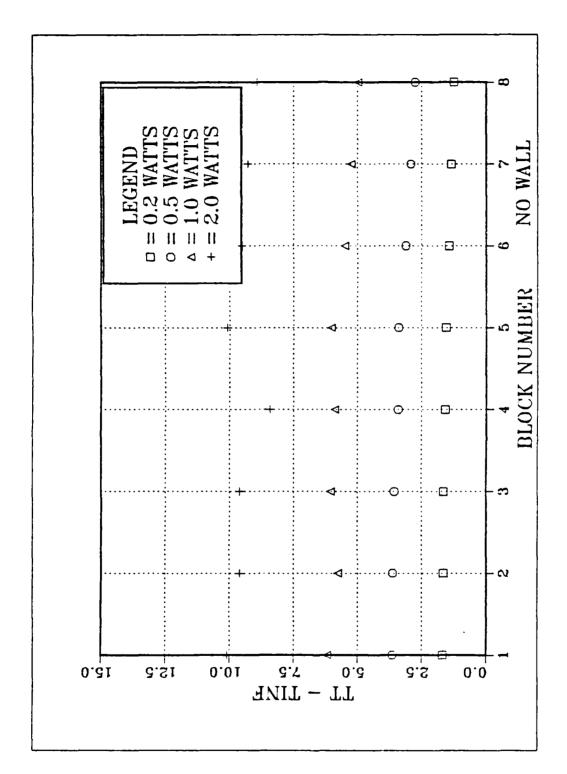


Block Number vs. Excess Temperature (Top Face) Runs 1-4 Figure 38.

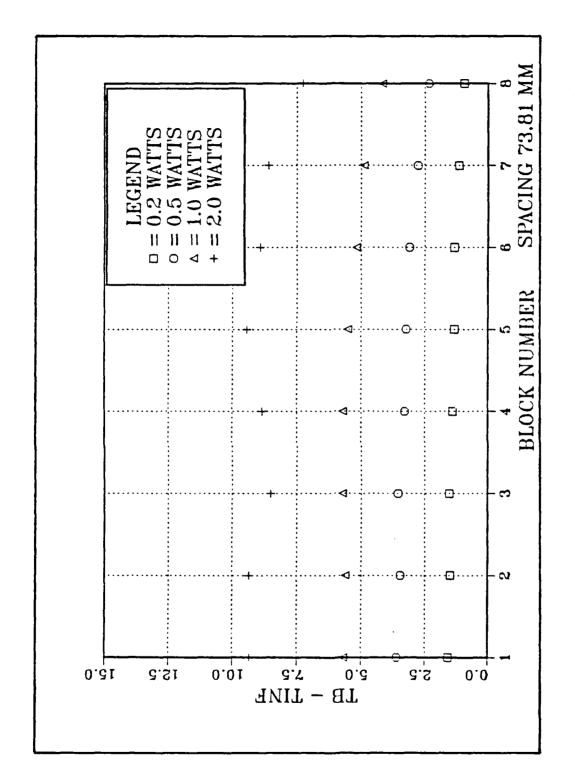


Block Number vs. Excess Temperature (Top Face) Runs 5-8 Figure 39.

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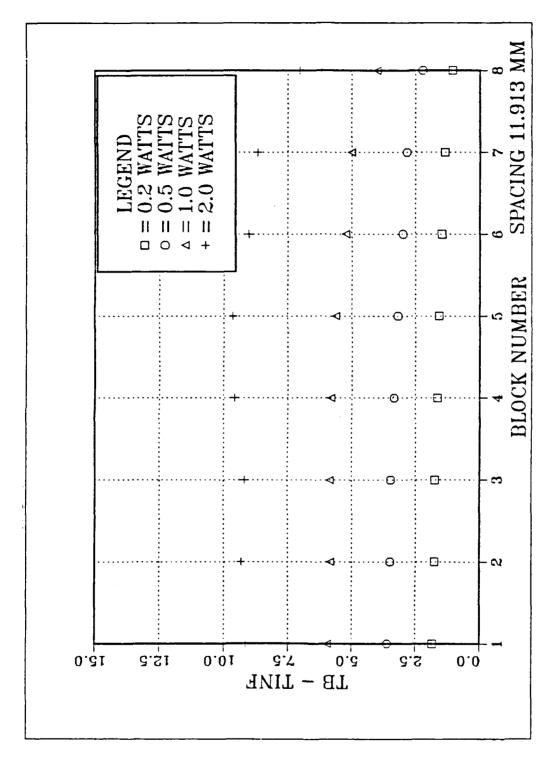


Block Number vs. Excess Temperature (Top Face) Runs 9-12 Figure 40.

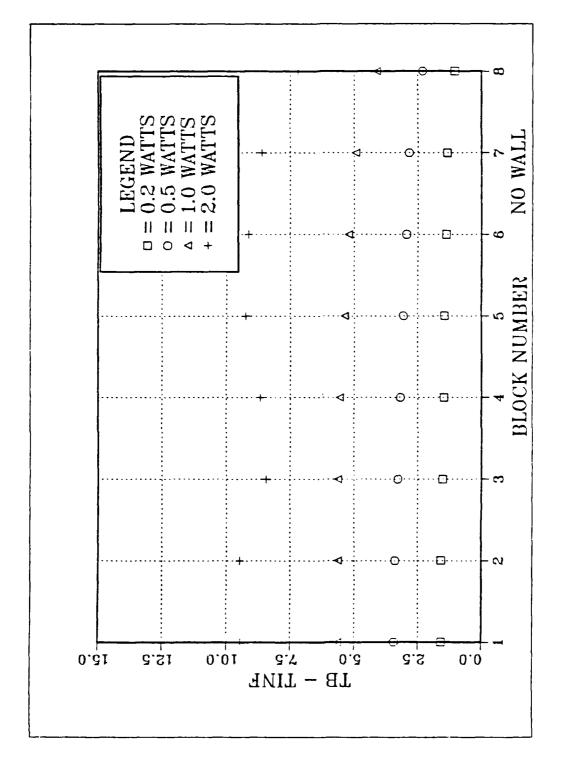


Block Number vs. Excess Temperature (Bottom Face) Runs 1-4 Figure 41.

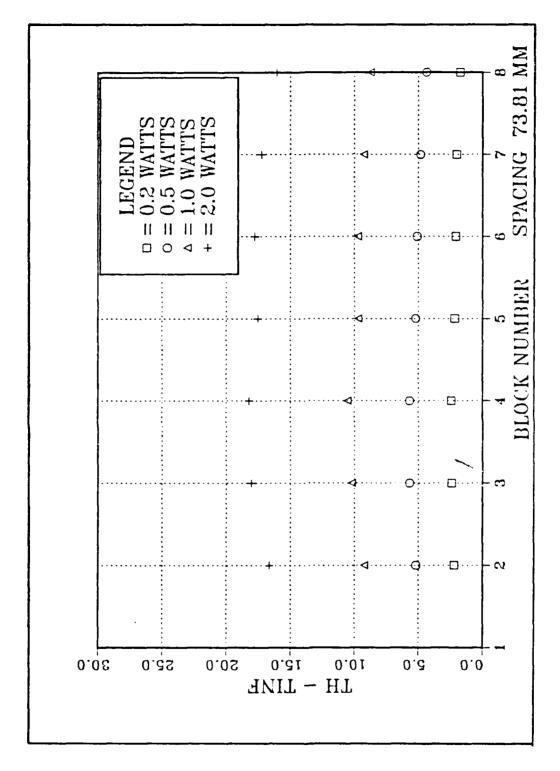
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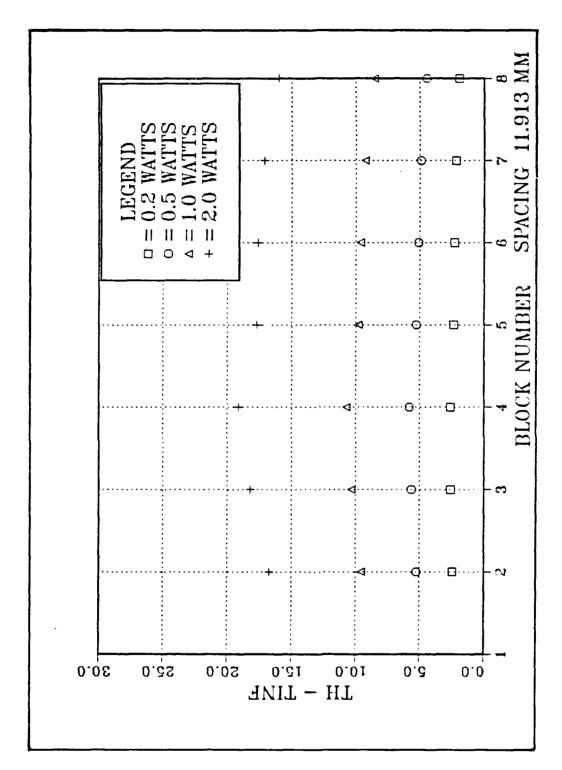
Block Number vs. Excess Temperature (Bottom Face) Runs 5-8 Figure 42.



Block Number vs. Excess Temperature (Bottom Face) Runs 9-12 Figure 43.

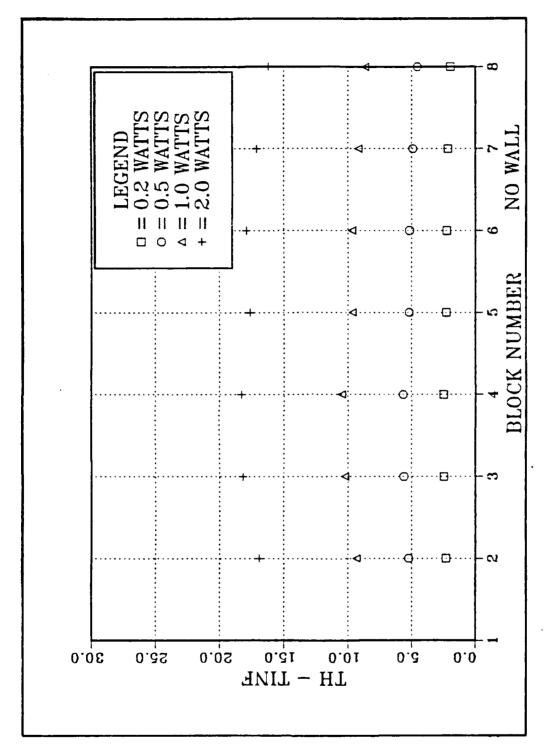


Block Number vs. Excess Temperature (Heater) Runs 1-4 Figure 44.

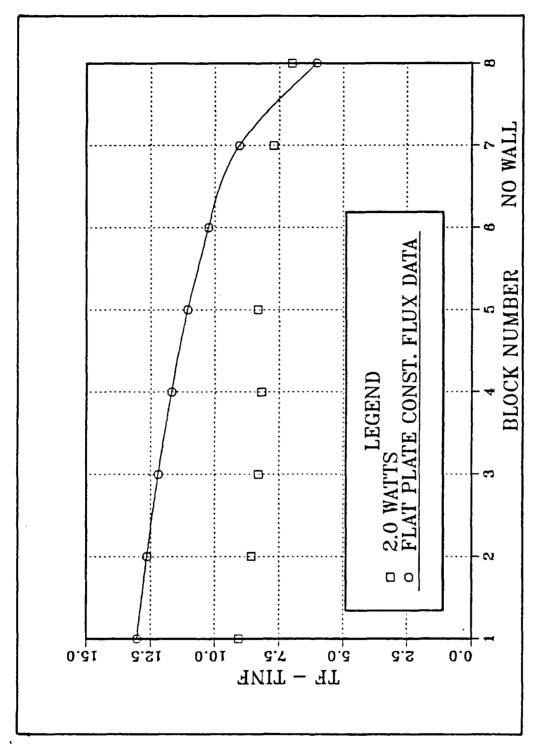


Block Number vs. Excess Temperature (Heater) Runs 5-8 Figure 45.

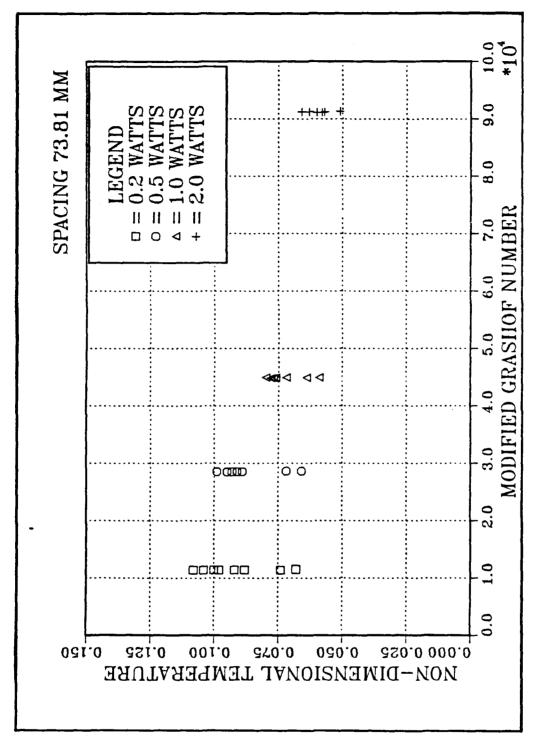
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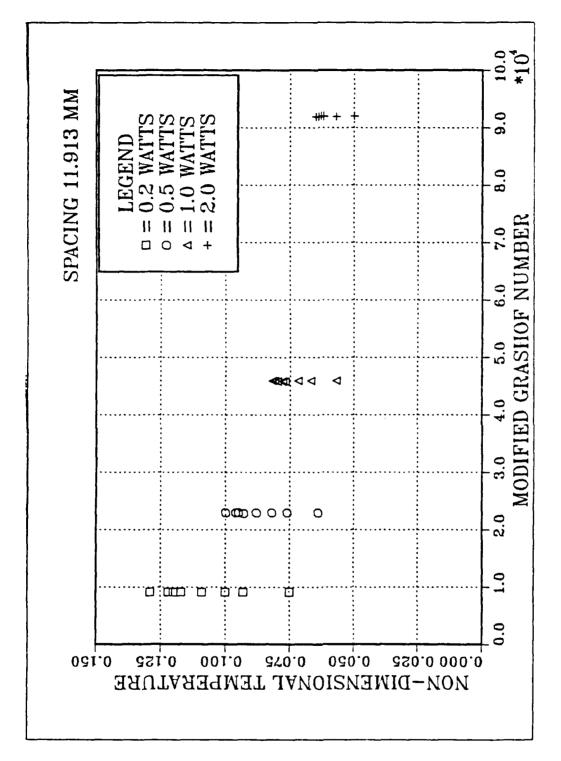
Block Number vs. Excess Temperature (Heater) Runs 9-12 Figure 46.



Block Number vs. Excess Temperature (Comparison of Front Face and a Flat Plate with Constant Heat Flux) Figure 47.

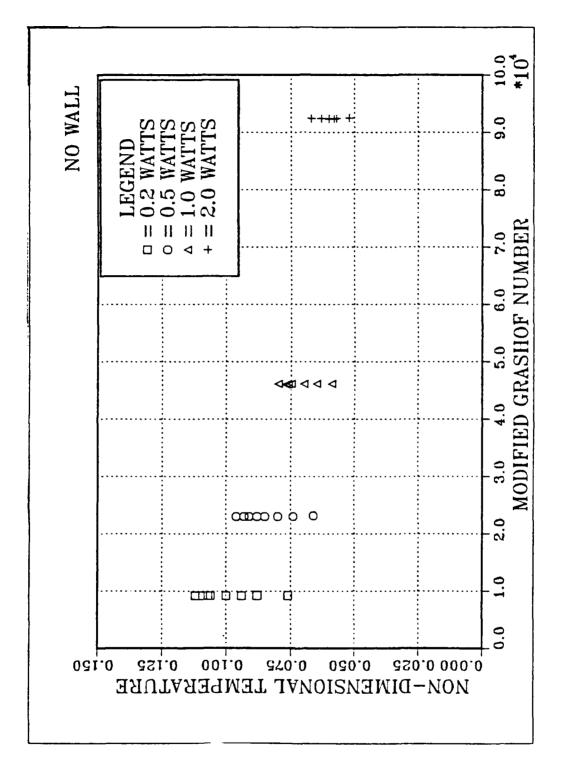


Modified Grashof Number vs. Nondimensional Temperature Runs 1-4 Figure 48.



Modified Grashof Number vs. Nondimensional Temperature Runs Figure 49.

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Modified Grashof Number vs. Nondimensional Temperature Runs 9-12 Figure 50.

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